

# FINITE ELEMENT METHOD

## ФИНИТ ЕЛЕМЕНТ МЕТОД

22

Differential  
Equation \*

# FINITE ELEMENT METHOD

## FINITE ELEMENT METHOD

STRONG FORM

Strong to Weak Form

WEAK FORM

Weak to Approximate Form

APPROXIMATE FORM

From Physical to Natural Space

NUMERICAL EVALUATION (Integration)

Approximate Solution to Differential Equation \*

ROADMAP

FOR FEM

1D  
2D

DISCRETIZED FORM

APPROXIMATION TECHNIQUES  
↳ SHAPE FUNCTIONS

# UNDERSTANDING FEM VIA AN ANALOGY (A BRUTAL SIMPLIFICATION)

Approximations in FEM

- Solution Approximation → inherent to numerical techniques
- Equation Approximation → diff equation is solved using computers
- Input Approximation → space transformed by discretization to weak form + space approximation



Discretization (Approximation) → DOMAIN ( $X$ )  
↓  
( $\rightarrow$  STRONG FORM)  
Solution ( $u$ )  
TEST ( $w$ )  
diff. Eq.  
integral TO  
 $\rightarrow$  WEAK FORM

# FINITE ELEMENT METHOD IN A NUTSHELL

Diff. Eq.  $\rightarrow$  2<sup>ND.</sup> O.D.E.

**STRONG FORM**

$$\int_0^L (EAu')' + b = 0$$

another source of approximation  $\rightarrow$  NUMERICAL INTEGRATION

**ELEMENT-WISE QUANTITIES**

PIECEWISE INTEGRALS (Solutions)

$\rightarrow$  (I) Multiply By  $w$   $\rightarrow$  (II) INTEGRATE

test function

Approximate Discretized Weak Form

**APPROXIMATE FORM**

**WEAK FORM**

INTEGRAL FORM

$$\int_0^1 w'u' dx = \int_0^1 w da$$

$$+ w(1)u'(1)$$

$$- w(0)u'(0)$$

PIECEWISE

**DISCRETIZED FORM**

Approximation

PostProcess

SOLVE

From GLOBAL TO ELEMENTS

From INTEGRAL OVER THE DOMAIN

ASSEMBLY

$$\int_0^1 \dots dx = \int_a^b \dots dx + \dots$$

$$[K][w] = [F]$$

# 1D FEM

## Overviews and Wrap-up

# From STRONG FORM TO ELEMENT STIFFNESS IN PHYSICAL SPACE now

$$EA u'' = 0 \quad \text{SUBJECT TO BCs}$$

  
 MULTIPLY BY  
 TEST  
 FUNCTION  
 $\omega$   


 } DIRICHLET  $\rightarrow u$  is PRESCRIBED  
 NEUMANN  $\rightarrow u'$  is PRESCRIBED



$$EA\omega u'' = 0 \quad \leftarrow \omega u'' = (\omega u')' - \omega u'$$

$$EA [(\omega u')' - \omega u'] = 0 \Rightarrow EA \omega' u' = EA (\omega u')' \quad \leftarrow \text{INTEGRATE}$$

$$\int_L EA \omega' u' dx = \int_L EA (\omega u')' dx = EA \omega u' \Big|_1^2 = EA \omega^2 u'^2 - EA \omega^1 u'^1$$

From STRONG FORM TO ELEMENT STIFFNESS  $\rightarrow$  IN PHYSICAL SPACE now

$$EA u'' = 0 \quad \text{SUBJECT TO BCs}$$



$$EA \begin{bmatrix} \int_L N^1' N^1' dx & \int_L N^1' N^2' dx \\ \int_L N^2' N^1' dx & \int_L N^2' N^2' dx \end{bmatrix} \begin{bmatrix} u^1 \\ u^2 \end{bmatrix} = \begin{bmatrix} F^1 \\ F^2 \end{bmatrix}$$

$$\begin{bmatrix} K^{11} & K^{12} \\ K^{21} & K^{22} \end{bmatrix} \begin{bmatrix} u^1 \\ u^2 \end{bmatrix} = \begin{bmatrix} F^1 \\ F^2 \end{bmatrix} \quad \text{and} \quad K^{ij} = EA \int_L N^i' N^j' dx$$

$$K^{ij} = EA \int_L n^i' n^j' dx \quad \xrightarrow{\text{PHYSICAL}} \text{RECALL:}$$

$$= EA \int_{-1}^1 \frac{\partial N^i}{\partial \xi} \frac{\partial N^j}{\partial \xi} \bar{J}^{-1} d\xi \quad \xrightarrow{\text{NATURAL}}$$

$$\int_{-1}^1 g(\xi) d\xi = \sum_{GP=1}^{GPE} g(\xi) \alpha_{GP}$$

$\leftarrow$  Loop over GP

$$= EA \sum_{GP=1}^{GPE} \left\{ \left[ \frac{\partial N^i}{\partial \xi} \quad \frac{\partial N^j}{\partial \xi} \quad \bar{J}^{-1} \right] \Big|_{GP} \times \alpha_{GP} \right\} \quad \vdots \quad \text{END}$$

)  
eg.

WHAT YOU  
SEE IN THE  
CODE !

{ For  $GP=1: GPE$   
in  
MATLAB  
End

# 2D FEM

Differential  
Equation \*

# FINITE ELEMENT METHOD

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From Physical to Natural Space

NUMERICAL EVALUATION (Integration)

Approximate Solution to Differential Equation \*

ROADMAP

FOR FEM

1D  
2D

DISCRETIZED FORM

APPROXIMATION TECHNIQUES  
↳ SHAPE FUNCTIONS

# MATHEMATICAL PRELIMINARIES

# EINSTEIN SUMMATION CONVENTION

A little definition for  
notation convenience

}

A REPEATED INDEX TWICE MEANS SUMMATION OVER THAT INDEX

also, called "dummy index"

$$\sum_{i=1}^3 u_i v_i = u_1 v_1 + u_2 v_2 + u_3 v_3 \equiv u_i v_i \quad i \text{ is summation index}$$

$i$ : free index

$$\sum_{\substack{j=1 \\ 1 \leq i \leq 3}}^{i=3} A_{ij} u_j \Rightarrow \begin{cases} i=1 \Rightarrow A_{11} u_1 + A_{12} u_2 + A_{13} u_3 \\ i=2 \Rightarrow A_{21} u_1 + A_{22} u_2 + A_{23} u_3 \\ i=3 \Rightarrow A_{31} u_1 + A_{32} u_2 + A_{33} u_3 \end{cases} \Rightarrow \begin{bmatrix} A_{11} & A_{12} & A_{13} \\ A_{21} & A_{22} & A_{23} \\ A_{31} & A_{32} & A_{33} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \end{bmatrix} = A_{ij} u_j$$

$j$ : summation index

$$u \cdot v = u_x v_x + u_y v_y + u_z v_z = u_1 v_1 + u_2 v_2 + u_3 v_3 = \sum_{i=1}^3 u_i v_i = u_i v_i$$

Dot Product ( $u$ ,  $v$ )  $\rightarrow$  SCALAR  $\leftarrow u_i v_i \rightarrow u \cdot v$

Double Dot Product ( $A$ ,  $B$ )  $\rightarrow$  SCALAR  $\leftarrow A_{ij} B_{ij} \rightarrow A \cdot B$

$u \otimes v$  Dyadic Product ( $u$ ,  $v$ )  $\rightarrow$  MATRIX  $\leftarrow u_i v_j \rightarrow [u \otimes v]_{ij}$

KRONECKER DELTA  $\rightarrow \delta_{ij} = \phi_i \cdot \phi_j \Rightarrow \begin{cases} 1 & i=j \\ 0 & i \neq j \end{cases}$

$$u_i \circ v_i = u_i \cdot v_i$$

$$\left[ A \cdot B \right]_{ik} = A_{ij} B_{jk}$$

$$\left[ u \otimes v \right]_{ij} = u_i \cdot v_j$$

$$\delta_{ij} = \phi_i \circ \phi_j$$

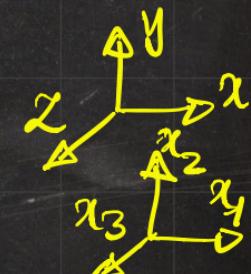
$$\left[ A \cdot u \right]_i = A_{ij} u_j$$

$$A \circ B = A_{ij} B_{ij}$$

$E$   $\rightsquigarrow$  FOURTH-ORDER TENSOR (ARRAY)  $\rightsquigarrow 3 \times 3 \times 3 \times 3 = 81$  Components

$\hookrightarrow$

2nd.      4th.      2nd.  
 $\nwarrow$        $\nearrow$        $\nwarrow$   
 $E = E \circ \Phi \rightarrow [E]_{ijk} = [E]_{ijkl} [\Phi]_{kl}$



INSTEAD OF  $x, y, z \rightarrow 1, 2, 3 \rightsquigarrow x_1, x_2, x_3$

$$\phi_x \circ \phi_1$$

DERIVATIVES  $\rightarrow$  e.g. STRONG FORM into "U"

$$y = f(x) \rightarrow y' = f'(x) \quad \text{and} \quad \{\cdot\}' = \frac{\partial \{\cdot\}}{\partial x}$$

$u = u(x, y, z)$  into GRAD  $u$  or DIV  $u$  or CURL  $u$

$$\left( \frac{\partial u}{\partial x_i} \otimes \phi_i \right)$$

$$\left( \frac{\partial u}{\partial x_i} \cdot \phi_i \right)$$

$$\left( \frac{\partial u}{\partial x_i} \times \phi_i \right)$$

DERIVATIVES e.g. STRONG FORM mo  $u''$

$$y = f(x) \rightarrow y' = f'(x) \quad \text{and} \quad \xi^i = \frac{\delta \xi^i}{\delta x} \rightarrow \text{GRAD } u = \frac{\partial u}{\partial x_i} \otimes \phi_i$$

$u = u(x, y, z)$  mo GRAD  $u$  or  $\text{Div } u$

$$u = f(x, y, z) \rightarrow \text{GRAD } f \quad \text{and} \quad \frac{\partial f}{\partial x_i} \phi_i \rightarrow \text{Div } u = \frac{\partial u}{\partial x_i} \cdot \phi_i$$

$A = A(x, y, z)$  mo GRAD  $A$  or  $\text{Dir } A$

$$\left( \frac{\partial A}{\partial x_i} \otimes \phi_i \right) \quad \left( \frac{\partial A}{\partial x_i} \cdot \phi_i \right)$$

SCALAR → 0, VECTOR → 1, MATRIX → 2

$$\text{GRAD } \phi = \begin{bmatrix} \frac{\partial \phi}{\partial x_1} \\ \frac{\partial \phi}{\partial x_2} \\ \frac{\partial \phi}{\partial x_3} \end{bmatrix}$$

$$\text{GRAD } u = \begin{bmatrix} \frac{\partial u_1}{\partial x_1} & \frac{\partial u_1}{\partial x_2} & \frac{\partial u_1}{\partial x_3} \\ \frac{\partial u_2}{\partial x_1} & \frac{\partial u_2}{\partial x_2} & \frac{\partial u_2}{\partial x_3} \\ \frac{\partial u_3}{\partial x_1} & \frac{\partial u_3}{\partial x_2} & \frac{\partial u_3}{\partial x_3} \end{bmatrix}$$

$$\text{Div } u = \frac{\partial u_1}{\partial x_1} + \frac{\partial u_2}{\partial x_2} + \frac{\partial u_3}{\partial x_3}$$

GRADIENT INCREASES  
THE ORDER BY 1

DIVERGENCE REDUCES  
THE ORDER BY 1

$$\text{Div } A = \begin{bmatrix} \frac{\partial A_{11}}{\partial x_1} + \frac{\partial A_{12}}{\partial x_2} + \frac{\partial A_{13}}{\partial x_3} \\ \frac{\partial A_{21}}{\partial x_1} + \frac{\partial A_{22}}{\partial x_2} + \frac{\partial A_{23}}{\partial x_3} \\ \frac{\partial A_{31}}{\partial x_1} + \frac{\partial A_{32}}{\partial x_2} + \frac{\partial A_{33}}{\partial x_3} \end{bmatrix}$$

# Big Picture of Mechanics (Mechanical Problems & Thermal Problems)

Def.  
 $u$



Load.  
 $t$



$$\nabla \cdot \sigma = \text{GRAD } u$$

$$\nabla \cdot \sigma = \text{GRAD } u$$

STRAIN  
 $\epsilon$



STRESS  
 $\sigma$

$$\sigma = C : \epsilon$$

$\angle$  Hooke's law

Temp.  
 $\theta$



Heat Flux  
 $q_n$

CHauchy

$$\nabla \cdot q = \text{GRAD } \theta$$

$$\nabla \cdot q = \text{GRAD } \theta$$

Temp.  
GRADIENT  
 $\nabla \theta$



Heat Flux  
vector  
 $q$

$$q = -K \cdot \nabla \theta$$

$\angle$  Fourier's law

STRONG FORM (GENERIC FORM)  $\rightarrow$   $\text{Div } \sigma_{ij} + b_i = 0$ ,  $\text{Div } q_i + c = 0$

$$\left\{ \begin{array}{l} \frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} + b_x = 0 \\ \frac{\partial \sigma_{yx}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} + b_y = 0 \end{array} \right.$$

$$\frac{\partial \sigma_{jk}}{\partial k} + b_j = 0$$

$$\frac{\partial q_i}{\partial x_i} + c = 0$$

$$\frac{\partial q_x}{\partial x} + \frac{\partial q_y}{\partial y} + c = 0$$

2D  $\rightarrow$  Plane STRAIN  
Plane STRESS

$$\left. \begin{array}{l} \sigma_{ij} = q_i \\ c = f \end{array} \right\}$$



# APPROXIMATION USING 2D FINITE ELEMENTS Shape Functions

EXAMPLE I:

$$f(-1, -1) = 1$$

$$f(1, -1) = 2$$

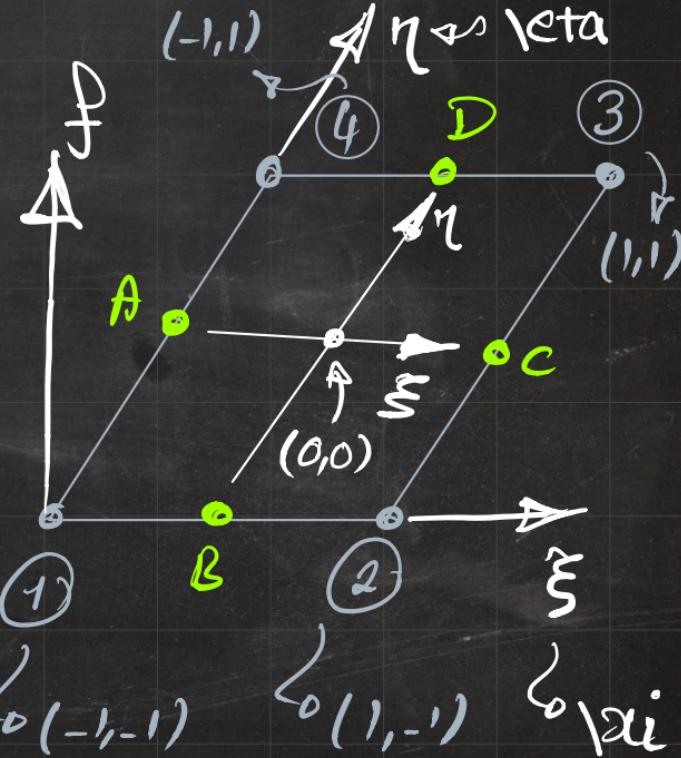
$$f(1, 1) = -1$$

$$f(-1, 1) = 4$$

$$\left\{ \begin{array}{l} f(0,0) = ? \\ , f(\xi, \eta) = ? \end{array} \right.$$

$$f(\xi, \eta) = N^1 f^1 + N^2 f^2 + N^3 f^3 + N^4 f^4$$

$$\left\{ \begin{array}{l} (-1, -1) \\ (1, -1) \end{array} \right.$$



# APPROXIMATION USING 2D FINITE ELEMENTS Shape Functions

EXAMPLE I:

$$f(\xi, \eta) = N^i f^i$$

$$= \frac{1}{4} [\xi - 1][\eta - 1] \times 1$$

$$- \frac{1}{4} [\xi + 1][\eta - 1] \times 2$$

$$+ \frac{1}{4} [\xi + 1][\eta + 1] \times -1$$

$$- \frac{1}{4} [\xi - 1][\eta + 1] \times 4$$

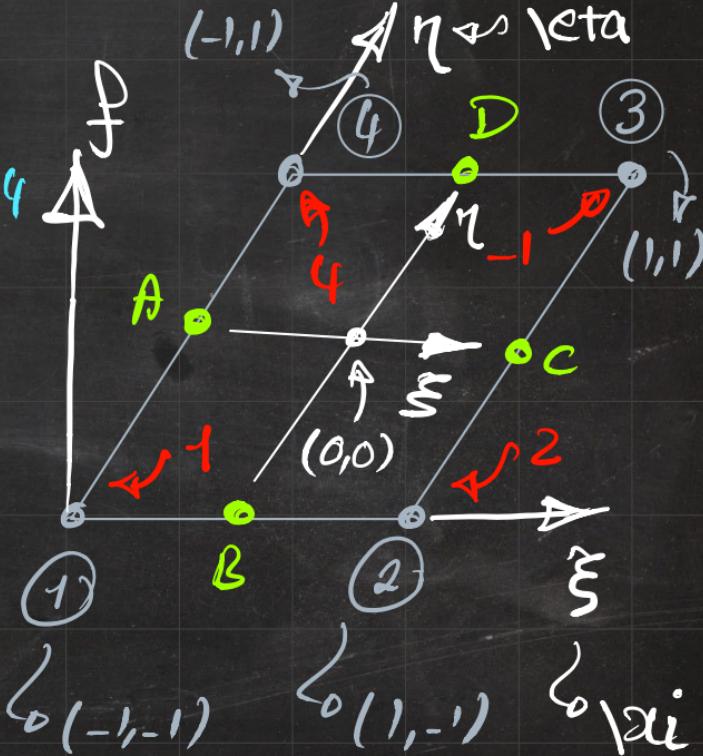
$$= 000 = f(\xi, \eta) \checkmark$$

$$N^1(\xi, \eta) = \frac{1}{4} [\xi - 1][\eta - 1]$$

$$N^2(\xi, \eta) = -\frac{1}{4} [\xi + 1][\eta - 1]$$

$$N^3(\xi, \eta) = \frac{1}{4} [\xi + 1][\eta + 1]$$

$$N^4(\xi, \eta) = -\frac{1}{4} [\xi - 1][\eta + 1]$$



# 2D Finite Element Library

## two-dimensional finite elements library

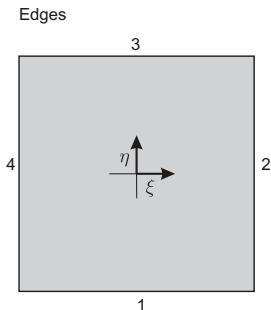
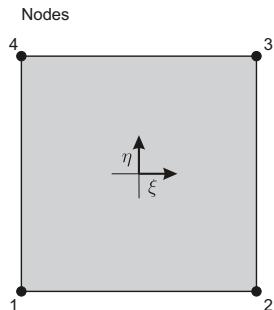


- two-dimensional 4-noded quadrilateral element (D2QU4N)
  - a.k.a. bilinear quadrilateral element
- two-dimensional 9-noded quadrilateral element (D2QU9N)
  - a.k.a. Lagrange biquadratic quadrilateral element
- two-dimensional 8-noded quadrilateral element (D2QU8N)
  - a.k.a. serendipity biquadratic quadrilateral element
- two-dimensional 3-noded triangular element (D2TR3N)
  - a.k.a. constant strain triangle
- two-dimensional 6-noded triangular element (D2TR6N)
  - a.k.a. quadratic triangle
- two-dimensional quadrature rule

# 2D Finite Element Library

D2QU4N

bilinear quadrilateral element



Node Number	Coordinates	
	$\xi$	$\eta$
1	-1	-1
2	1	-1
3	1	1
4	-1	1

$$N^1 = \frac{1}{4} (1 - \xi) (1 - \eta)$$

$$N_{,\xi}^1 = -\frac{1}{4} (1 - \eta) \quad N_{,\eta}^1 = -\frac{1}{4} (1 - \xi)$$

$$N^2 = \frac{1}{4} (1 + \xi) (1 - \eta)$$

$$N_{,\xi}^2 = +\frac{1}{4} (1 - \eta) \quad N_{,\eta}^2 = -\frac{1}{4} (1 + \xi)$$

$$N^3 = \frac{1}{4} (1 + \xi) (1 + \eta)$$

$$N_{,\xi}^3 = +\frac{1}{4} (1 + \eta) \quad N_{,\eta}^3 = +\frac{1}{4} (1 + \xi)$$

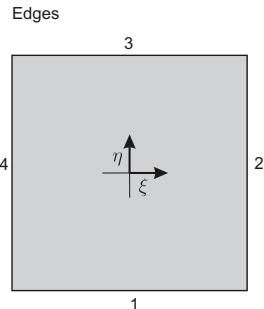
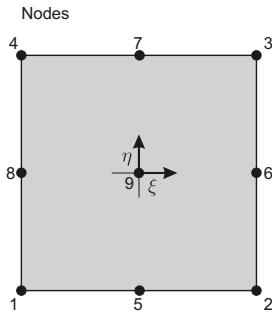
$$N^4 = \frac{1}{4} (1 - \xi) (1 + \eta)$$

$$N_{,\xi}^4 = -\frac{1}{4} (1 + \eta) \quad N_{,\eta}^4 = +\frac{1}{4} (1 - \xi)$$

# 2D Finite Element Library

D2QU9N

Lagrange biquadratic quadrilateral element



Node Number	Coordinates	
	$\xi$	$\eta$
1	-1	-1
2	1	-1
3	1	1
4	-1	1
5	0	-1
6	1	0
7	0	1
8	-1	0
9	0	0

$$N^1 = +\frac{1}{4} (1 - \xi) \xi (1 - \eta) \eta$$

$$N^2 = -\frac{1}{4} (1 + \xi) \xi (1 - \eta) \eta$$

$$N^3 = +\frac{1}{4} (1 + \xi) \xi (1 + \eta) \eta$$

$$N^4 = -\frac{1}{4} (1 - \xi) \xi (1 + \eta) \eta$$

$$N^5 = -\frac{1}{2} (1 - \xi) (1 + \xi) (1 - \eta) \eta$$

$$N^6 = +\frac{1}{2} (1 + \xi) \xi (1 - \eta) (1 + \eta)$$

$$N^7 = +\frac{1}{2} (1 - \xi) (1 + \xi) (1 + \eta) \eta$$

$$N^8 = -\frac{1}{2} (1 - \xi) \xi (1 - \eta) (1 + \eta)$$

$$N^9 = (1 - \xi) (1 + \xi) (1 - \eta) (1 + \eta)$$

$$N_{,\xi}^1 = +\frac{1}{4} (1 - 2\xi) (1 - \eta) \eta$$

$$N_{,\xi}^2 = -\frac{1}{4} (1 + 2\xi) (1 - \eta) \eta$$

$$N_{,\xi}^3 = +\frac{1}{4} (1 + 2\xi) (1 + \eta) \eta$$

$$N_{,\xi}^4 = -\frac{1}{4} (1 - 2\xi) (1 + \eta) \eta$$

$$N_{,\xi}^5 = \xi \eta (1 - \eta)$$

$$N_{,\xi}^6 = \frac{1}{2} (1 + 2\xi) (1 - \eta) (1 + \eta)$$

$$N_{,\xi}^7 = -\xi \eta (1 + \eta)$$

$$N_{,\xi}^8 = -\frac{1}{2} (1 - 2\xi) (1 - \eta) (1 + \eta)$$

$$N_{,\xi}^9 = -2\xi (1 - \eta) (1 + \eta)$$

$$N_{,\eta}^1 = +\frac{1}{4} (1 - \xi) \xi (1 - 2\eta)$$

$$N_{,\eta}^2 = -\frac{1}{4} (1 + \xi) \xi (1 - 2\eta)$$

$$N_{,\eta}^3 = +\frac{1}{4} (1 + \xi) \xi (1 + 2\eta)$$

$$N_{,\eta}^4 = -\frac{1}{4} (1 - \xi) \xi (1 + 2\eta)$$

$$N_{,\eta}^5 = \frac{1}{2} (1 - \xi) (1 + \xi) (2\eta - 1)$$

$$N_{,\eta}^6 = - (1 + \xi) \xi \eta$$

$$N_{,\eta}^7 = \frac{1}{2} (1 - \xi) (1 + \xi) (1 + 2\eta)$$

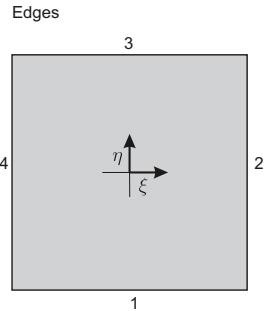
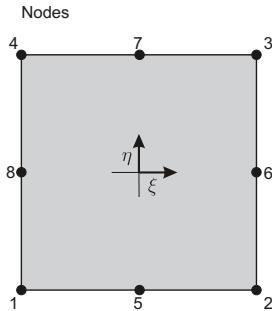
$$N_{,\eta}^8 = (1 - \xi) \xi \eta$$

$$N_{,\eta}^9 = -2 (1 - \xi) (1 + \xi) \eta$$

# 2D Finite Element Library

D2QU8N

serendipity biquadratic quadrilateral element



Node Number	Coordinates	
	$\xi$	$\eta$
1	-1	-1
2	1	-1
3	1	1
4	-1	1
5	0	-1
6	1	0
7	0	1
8	-1	0

$$N^1 = -\frac{1}{4} (1 - \xi) (1 - \eta) (1 + \xi + \eta)$$

$$N_{,\xi}^1 = +\frac{1}{4} (1 - \eta) (2\xi + \eta)$$

$$N_{,\eta}^1 = +\frac{1}{4} (1 - \xi) (\xi + 2\eta)$$

$$N^2 = -\frac{1}{4} (1 + \xi) (1 - \eta) (1 - \xi + \eta)$$

$$N_{,\xi}^2 = +\frac{1}{4} (1 - \eta) (2\xi - \eta)$$

$$N_{,\eta}^2 = +\frac{1}{4} (1 + \xi) (-\xi + 2\eta)$$

$$N^3 = -\frac{1}{4} (1 + \xi) (1 + \eta) (1 - \xi - \eta)$$

$$N_{,\xi}^3 = +\frac{1}{4} (1 + \eta) (2\xi + \eta)$$

$$N_{,\eta}^3 = +\frac{1}{4} (1 + \xi) (\xi + 2\eta)$$

$$N^4 = -\frac{1}{4} (1 - \xi) (1 + \eta) (1 + \xi - \eta)$$

$$N_{,\xi}^4 = +\frac{1}{4} (1 + \eta) (2\xi - \eta)$$

$$N_{,\eta}^4 = +\frac{1}{4} (1 - \xi) (-\xi + 2\eta)$$

$$N^5 = \frac{1}{2} (1 - \xi) (1 + \xi) (1 - \eta)$$

$$N_{,\xi}^5 = -\xi (1 - \eta)$$

$$N_{,\eta}^5 = -\frac{1}{2} (1 - \xi) (1 + \xi)$$

$$N^6 = \frac{1}{2} (1 + \xi) (1 + \eta) (1 - \eta)$$

$$N_{,\xi}^6 = +\frac{1}{2} (1 - \eta) (1 + \eta)$$

$$N_{,\eta}^6 = -(1 + \xi) \eta$$

$$N^7 = \frac{1}{2} (1 - \xi) (1 + \xi) (1 + \eta)$$

$$N_{,\xi}^7 = -\xi (1 + \eta)$$

$$N_{,\eta}^7 = +\frac{1}{2} (1 - \xi) (1 + \xi)$$

$$N^8 = \frac{1}{2} (1 - \xi) (1 + \eta) (1 - \eta)$$

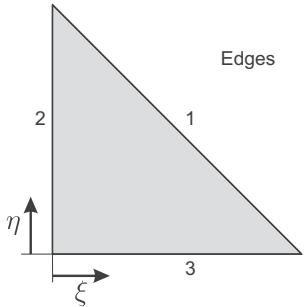
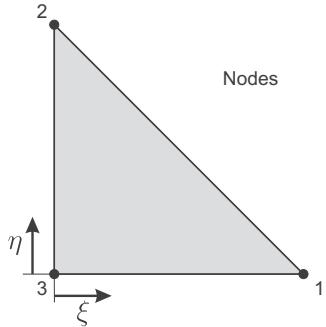
$$N_{,\xi}^8 = -\frac{1}{2} (1 - \eta) (1 + \eta)$$

$$N_{,\eta}^8 = -(1 - \xi) \eta$$

# 2D Finite Element Library

D2TR3N

constant strain triangle (CST)



Node Number	Coordinates	
	$\xi$	$\eta$
1	1	0
2	0	1
3	0	0

$$N^1 = \xi$$

$$N_{,\xi}^1 = 1$$

$$N_{,\eta}^1 = 0$$

$$N^2 = \eta$$

$$N_{,\xi}^2 = 0$$

$$N_{,\eta}^2 = 1$$

$$N^3 = (1 - \xi - \eta)$$

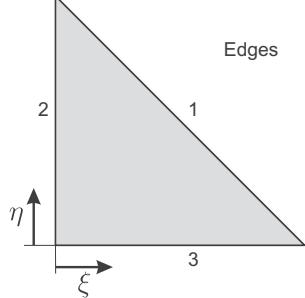
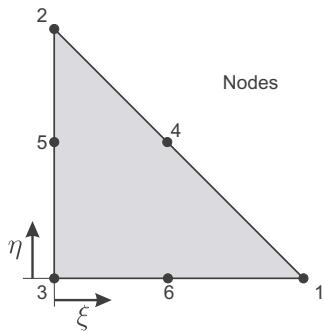
$$N_{,\xi}^3 = -1$$

$$N_{,\eta}^3(\xi, \eta) = -1$$

# 2D Finite Element Library

D2TR6N

quadratic triangle



Node Number	Coordinates	
	$\xi$	$\eta$
1	1	0
2	0	1
3	0	0
4	1/2	1/2
5	0	1/2
6	1/2	0

$$N^1 = \xi(2\xi - 1)$$

$$N_{,\xi}^1 = -1 + 4\xi$$

$$N_{,\eta}^1 = 0$$

$$N^2 = \eta(2\eta - 1)$$

$$N_{,\xi}^2 = 0$$

$$N_{,\eta}^2 = -1 + 4\eta$$

$$N^3 = (1 - \xi - \eta)(1 - 2\xi - 2\eta)$$

$$N_{,\xi}^3 = -3 + 4\xi + 4\eta$$

$$N_{,\eta}^3 = -3 + 4\xi + 4\eta$$

$$N^4 = 4\xi\eta$$

$$N_{,\xi}^4 = 4\eta$$

$$N_{,\eta}^4 = 4\xi$$

$$N^5 = 4\eta(1 - \xi - \eta)$$

$$N_{,\xi}^5 = -4\eta$$

$$N_{,\eta}^5 = -4(-1 + 2\eta + \xi)$$

$$N^6 = 4\xi(1 - \xi - \eta)$$

$$N_{,\xi}^6 = -4(-1 + \eta + 2\xi)$$

$$N_{,\eta}^6 = -4\xi$$

# 2D Finite Element Library

## two-dimensional quadrature rule i

### Triangular Elements Gauss Point Rule

$$\int_0^1 \int_0^{1-\eta} \{\bullet\} \, d\xi \, d\eta \approx \frac{1}{2} \sum_{i=1}^{N_{\text{Gauss Points}}} \alpha_i \, \{\bullet\}|_{\text{Gauss Point}^i}$$

Gauss Point Number	Coordinates		Weight Factor
	$\xi$	$\eta$	
1	1/3	1/3	1

Gauss Point Number	Coordinates		Weight Factor
	$\xi$	$\eta$	
1	1/6	1/6	1/3
2	4/6	1/6	1/3
3	1/6	4/6	1/3

# 2D Finite Element Library

## two-dimensional quadrature rule ii

### Quadrilateral Elements Gauss Point Rule

$$\int_{-1}^1 \int_{-1}^1 \{\bullet\} \, d\xi \, d\eta \approx \sum_{i=1}^{N_{\text{Gauss Points}}} \alpha_i \, \{\bullet\}|_{\text{Gauss Point } i}$$

Gauss Point Number	Coordinates		Weight Factor
	$\xi$	$\eta$	
1	0	0	$2 \times 2$

Gauss Point Number	Coordinates		Weight Factor
	$\xi$	$\eta$	
1	$-1/\sqrt{3}$	$-1/\sqrt{3}$	$1 \times 1$
2	$+1/\sqrt{3}$	$-1/\sqrt{3}$	$1 \times 1$
3	$+1/\sqrt{3}$	$+1/\sqrt{3}$	$1 \times 1$
4	$-1/\sqrt{3}$	$+1/\sqrt{3}$	$1 \times 1$

# 2D Finite Element Library

## two-dimensional quadrature rule iii

Gauss Point Number	Coordinates		Weight Factor
	$\xi$	$\eta$	$\alpha$
1	$-\sqrt{3/5}$	$-\sqrt{3/5}$	$5/9 \times 5/9$
2	$+\sqrt{3/5}$	$-\sqrt{3/5}$	$5/9 \times 5/9$
3	$\sqrt{3/5}$	$\sqrt{3/5}$	$5/9 \times 5/9$
4	$-\sqrt{3/5}$	$\sqrt{3/5}$	$5/9 \times 5/9$
5	0	$-\sqrt{3/5}$	$5/9 \times 8/9$
6	$+\sqrt{3/5}$	0	$5/9 \times 8/9$
7	0	$\sqrt{3/5}$	$5/9 \times 8/9$
8	$-\sqrt{3/5}$	0	$5/9 \times 8/9$
9	0	0	$8/9 \times 8/9$

Differential  
Equation \*

# FINITE ELEMENT METHOD

## FINITE ELEMENT METHOD

STRONG FORM

Strong to Weak Form

WEAK FORM

Weak to Approximate Form

APPROXIMATE FORM

From Physical to Natural Space

NUMERICAL EVALUATION (Integration)

Approximate Solution to Differential Equation \*

ROADMAP

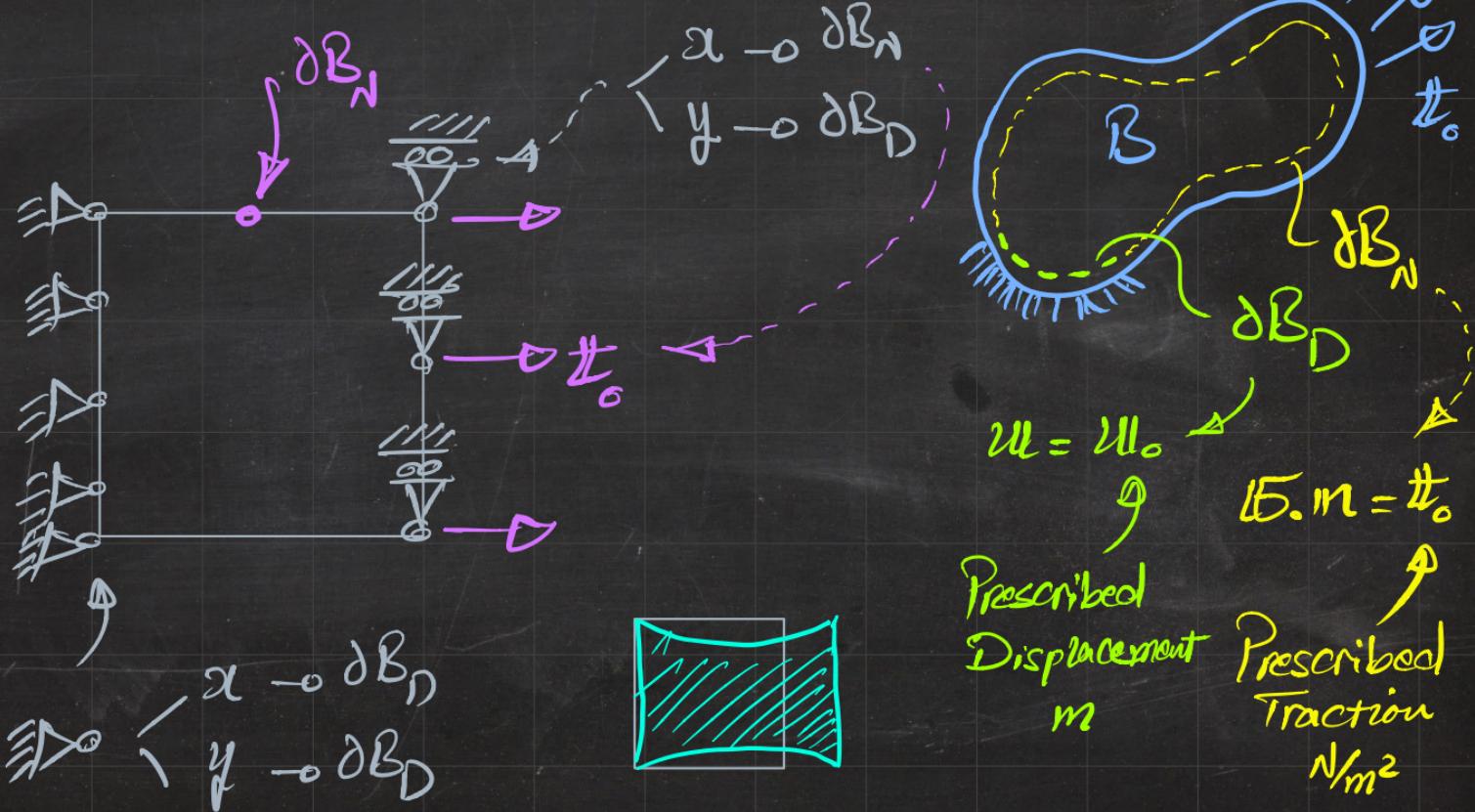
FOR FEM

1D  
2D

DISCRETIZED FORM

APPROXIMATION TECHNIQUES  
↳ SHAPE FUNCTIONS

# From STRONG FORM TO WEAK FORM



# From STRONG FORM TO WEAK FORM

$\operatorname{Div} \boldsymbol{\sigma} = \phi$  in  $B$  subject to BCs

STRONG FORM IN THE  
ABSENCE OF BODY FORCES

dot

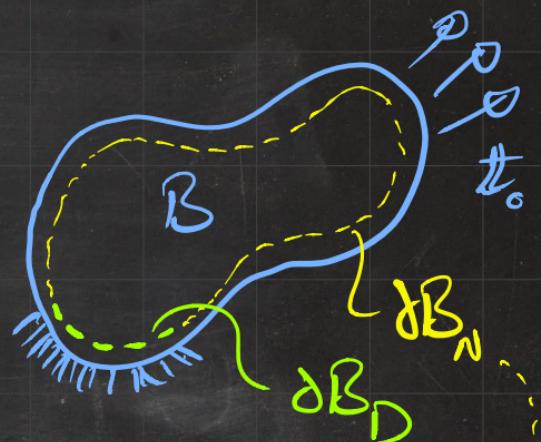
w.  $\operatorname{Div} \boldsymbol{\sigma} = 0$  SCALAR

TEST FUNCTION

$\psi$

$$\psi / \partial B_D = \phi$$

$$\begin{cases} u = u_0 & \text{at } \partial B_D \\ \boldsymbol{\sigma} \cdot \mathbf{n} = t_0 & \text{at } \partial B_N \end{cases}$$



$$\begin{cases} u = u_0 \\ \boldsymbol{\sigma} \cdot \mathbf{n} = t_0 \end{cases}$$

Prescribed Displacement  $m$  Prescribed Traction  $N/m^2$

# From STRONG FORM TO WEAK FORM

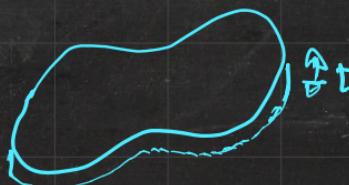
$\text{Div } \mathbf{B} = \phi$  in  $B$  subject to BCs

$$\omega \cdot \text{Div } \mathbf{B} = 0$$

$$\omega \cdot \text{Div } \mathbf{B} = \text{Div}(\omega \cdot \mathbf{B}) - [\text{GRAD } \omega] : \mathbf{B}$$

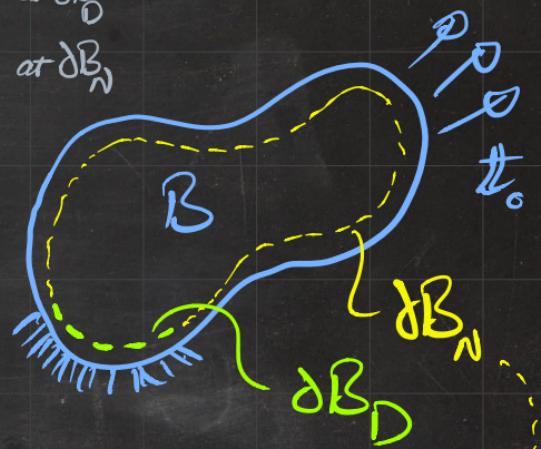
$$\int_B [\text{GRAD } \omega] : \mathbf{B} dV = \int_B \text{Div}(\omega \cdot \mathbf{B}) dV$$

$t dA$



$$u = u_0 \text{ at } \delta B_D$$

$$B \cdot n = t_0 \text{ at } \delta B_N$$



$$u = u_0$$

$$B \cdot n = t_0$$

Prescribed Displacement  $m$  Prescribed Traction  $N/m^2$

# From STRONG FORM TO WEAK FORM

$\text{Div } \mathbf{B} = \phi$  in  $B$  subject to BCs

$$w \cdot \text{Div } \mathbf{B} = 0$$

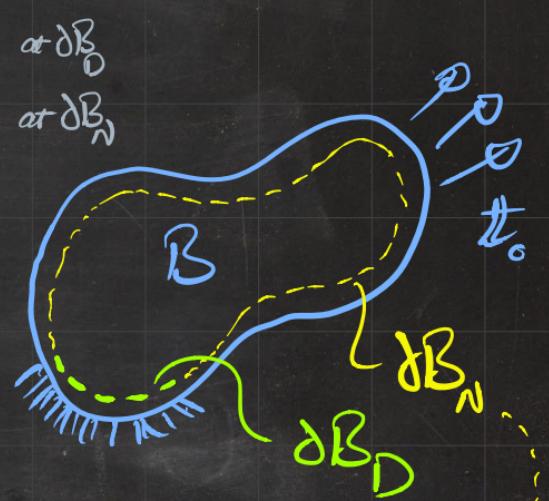
$$\int_B [\text{Grad } w] : \mathbf{B} \, dA = \int_B \text{Div}(w \cdot \mathbf{B}) \, dA$$

Divergence Theorem



$$\int_B \text{Div } \xi_0 \varphi \, dA = \int_{\partial B} \xi_0 \varphi \cdot \mathbf{n} \, dL$$

unit  
outward  
normal  
vector  
to boundary



$$w = w_0$$

Prescribed  
Displacement

$$w$$

$$B \cdot n = t_0$$

Prescribed  
Traction

$$N/m^2$$

# From STRONG FORM TO WEAK FORM

$\text{Div } \mathbf{B} = \phi$  in  $B$  subject to BCs

$$\mathbf{w} \cdot \text{Div } \mathbf{B} = 0$$

$$\int_B [\text{Grad } \mathbf{w}] : \mathbf{B} \, dA = \int_B \text{Div}(\mathbf{w} \cdot \mathbf{B}) \, dA$$

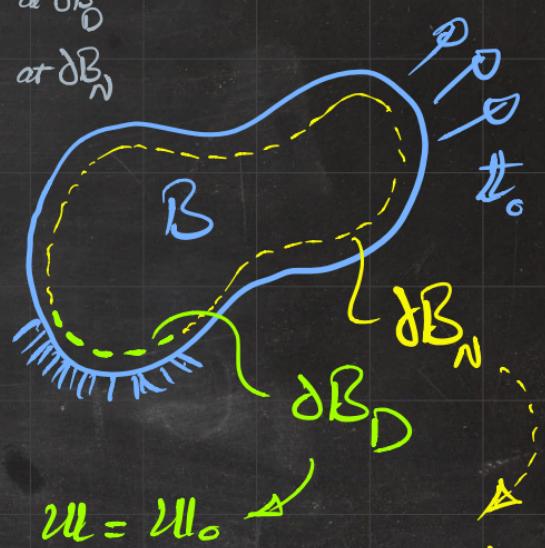
$$\int_B [\text{Grad } \mathbf{w}] : \mathbf{B} \, dA = \int_{\partial B} \mathbf{w} \cdot \mathbf{B} \cdot \mathbf{n} \, dL$$

$$\partial B_D \cap \partial B_N = \emptyset$$

$$\partial B = \partial B_D \cup \partial B_N$$

$$\mathbf{u} = \mathbf{u}_0 \text{ at } \partial B_D$$

$$\mathbf{B} \cdot \mathbf{n} = t_0 \text{ at } \partial B_N$$



$$\mathbf{u} = \mathbf{u}_0$$

Prescribed  
Displacement

$m$

$$\mathbf{B} \cdot \mathbf{n} = t_0$$

Prescribed  
Traction

$N/m^2$

# From STRONG FORM TO WEAK FORM

$\text{Div } \mathbf{B} = \phi$  in  $B$  subject to BCs

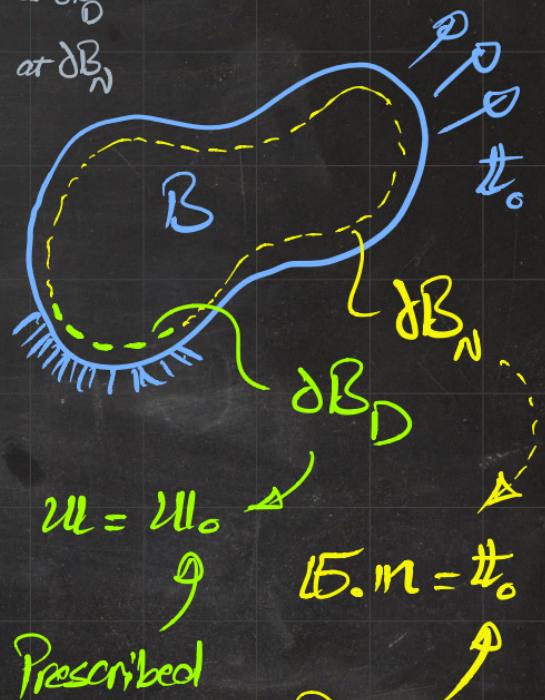
$$\mathbf{w} \cdot \text{Div } \mathbf{B} = 0 \quad \forall \omega, \omega|_{\partial B_D} = \phi$$

$$\int_B [\text{Grad } \omega] : \mathbf{B} \, dA = \int_B \text{Div}(\mathbf{w} \cdot \mathbf{B}) \, dA$$

$$\int_B [\text{Grad } \omega] : \mathbf{B} \, dA = \int_{\partial B} \mathbf{w} \cdot \mathbf{B} \cdot \mathbf{n} \, dL$$

$$= \underbrace{\int_{\partial B_D} \mathbf{w} \cdot \mathbf{B} \cdot \mathbf{n} \, dL}_{m} + \underbrace{\int_{\partial B_N} \mathbf{w} \cdot \mathbf{B} \cdot \mathbf{n} \, dL}_{t_o}$$

$$\begin{aligned} \mathbf{u} &= \mathbf{u}_0 \text{ at } \partial B_D \\ \mathbf{B} \cdot \mathbf{n} &= t_o \text{ at } \partial B_N \end{aligned}$$



Prescribed Displacement  $m$  Prescribed Traction  $t_o$   $N/m^2$

# From STRONG FORM TO WEAK FORM

$\text{Div } \mathbf{B} = \phi$  in  $B$  subject to BCs

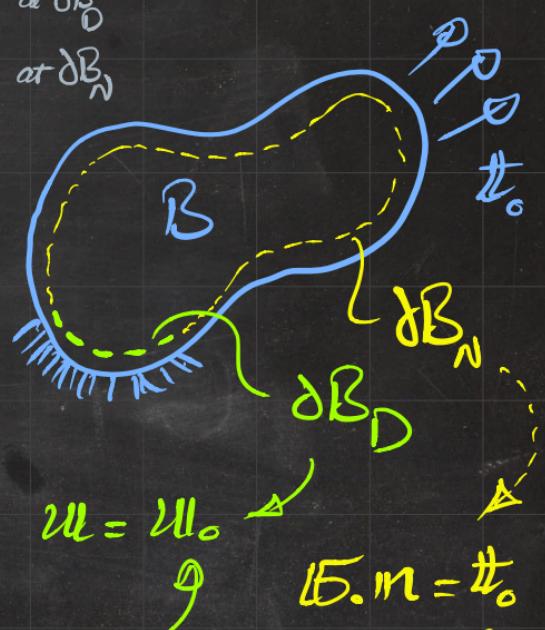
$w \cdot \text{Div } \mathbf{B} = 0 \quad \forall w, w|_{\partial B_D} = \phi$

$$\int_B [\text{GRAD } w] : \mathbf{B} \, dA = \int_B \text{Div}(w \cdot \mathbf{B}) \, dA$$

$$\boxed{\int_B [\text{GRAD } w] : \mathbf{B} \, dA = \int_{\partial B_N} w \cdot t_o \, dL}$$

$$w = w_o \text{ at } \partial B_D$$

$$\mathbf{B} \cdot \mathbf{n} = t_o \text{ at } \partial B_N$$



$$w = w_o$$

$$\mathbf{B} \cdot \mathbf{n} = t_o$$

Prescribed Displacement  $w$  Prescribed Traction  $t_o$   
 $N/m^2$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD \psi] : \underline{\sigma} dA = \int_{\partial B_N} \psi \cdot \underline{t}_o dL$$

$$\underline{\psi}^j \otimes GRAD \psi^j : \underline{\sigma} = E : \underline{\epsilon}$$

MAJOR  
MINOR

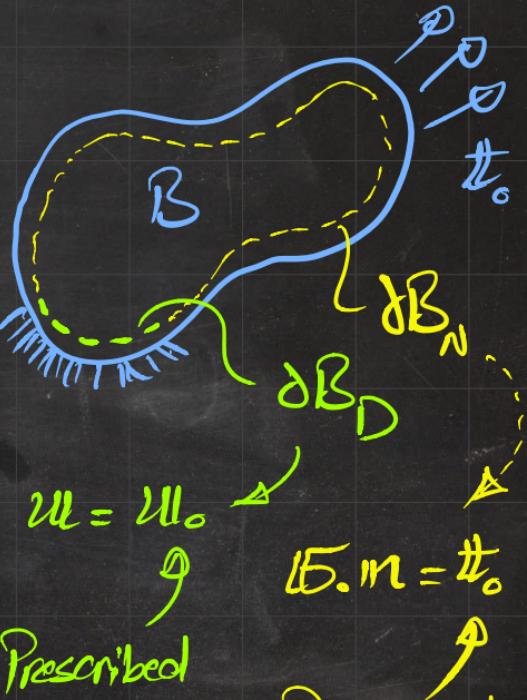
SYMMETRY

$$\underline{\epsilon} = GRAD \underline{u}^{sym}$$

$$\therefore \underline{\sigma} = E : [GRAD \underline{u}]$$

$$\psi = N^j \psi^j = N^1 \psi^1 + N^2 \psi^2 + \dots$$

$$GRAD \psi = \underline{\psi}^j \otimes GRAD \psi^j$$



Prescribed Displacement  $u$   
Prescribed Traction  $t_o$

$N/m^2$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD \omega] : \underline{\sigma} dA = \int_{\partial B_N} \omega \cdot \underline{t}_o dL$$

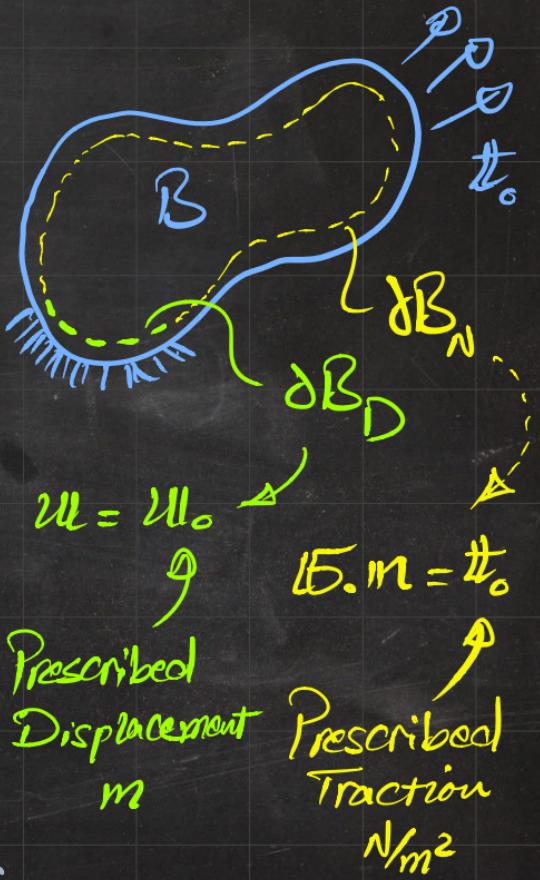
$$\underline{\sigma}^j \otimes GRAD N^j \quad \underline{\sigma} = E : \underline{\epsilon}$$

$$\underline{\sigma} = E : [GRAD \underline{u}]$$

$$GRAD \underline{u} = \underline{u}^i \otimes GRAD N^i$$

$$\underline{u} = N^i \underline{u}^i$$

$$= N^1 \underline{u}^1 + N^2 \underline{u}^2 + \dots$$



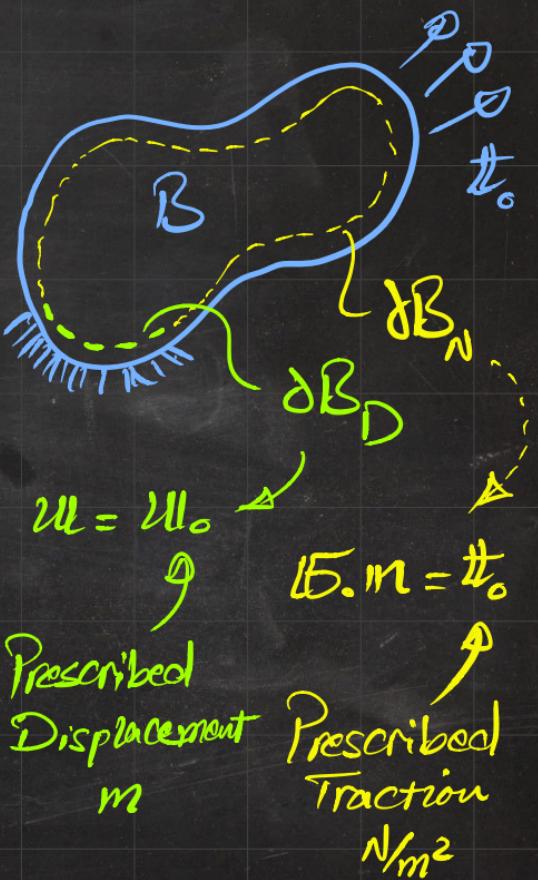
# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD \omega] : \underline{\sigma} dA = \int_{\partial B_N} \omega \cdot \underline{t}_o dL$$

$$\underline{\omega}^j \otimes GRAD N^j \quad \underline{\sigma} = E : \underline{\epsilon} \quad \omega = N^j \underline{\epsilon}^j$$

$$\underline{\sigma} = E : [GRAD \underline{u}] \quad GRAD \omega = \underline{\omega}^j \otimes GRAD N^j$$

$$\underline{\sigma} = E : [\underline{u}^i \otimes GRAD N^i]$$



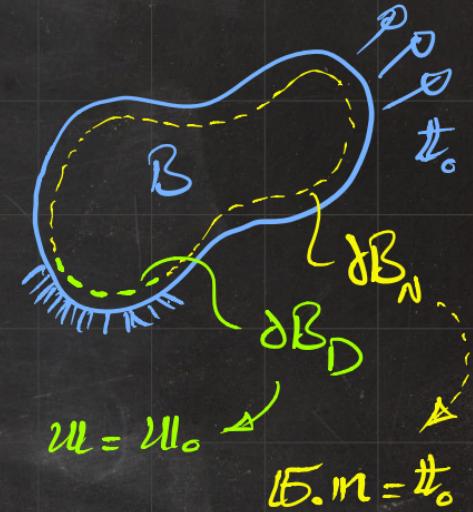
Prescribed Displacement  $u$   $\underline{u} = u_0$   
Prescribed Traction  $\sigma \cdot n$   $\underline{\sigma} \cdot \underline{n} = t_0$   
 $N/m^2$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD \psi] : \underline{\underline{B}} dA = \int_{\partial B_N} \psi \cdot \underline{t}_o dL$$

$$\underline{\psi}^j \otimes GRAD N^j \quad \underline{\underline{B}} = E : \underline{\underline{\epsilon}} \quad \psi = N^j \underline{\psi}^j$$

$$\underline{\underline{B}} = E : [\underline{\psi}^i \otimes GRAD N^i] \quad GRAD \psi = \underline{\psi}^j \otimes GRAD N^j$$



$$GRAD \xi_o \vec{r} = \xi_o \vec{r}_{,x}$$

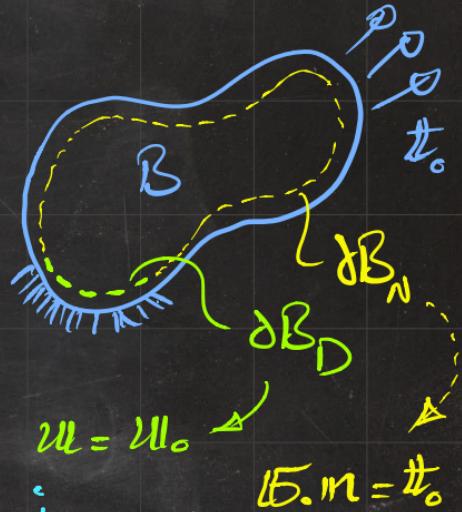
$$\int_B [\underline{\psi}^j \otimes N^j_{,x}] : E : [\underline{\psi}^i \otimes N^i_{,x}] dA = \int_{\partial B_N} N^j \underline{\psi}^j \cdot \underline{t}_o dL$$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [ \text{GRAD}(\mathbf{w}) ] : \mathbf{B} dA = \int_{\partial B_N} \mathbf{w} \cdot \mathbf{t}_o dL$$

$$\int_B [\mathbf{w}^j \otimes \mathbf{N}_{,2i}^j] : \mathbf{E} : [\mathbf{u}^i \otimes \mathbf{N}_{,2i}] dA = \int_{\partial B_N} \mathbf{N}^j \mathbf{w}^j \cdot \mathbf{t}_o dL$$

$E_{abcd} \quad \Phi_a \otimes \Phi_b \otimes \Phi_c \otimes \Phi_d$



$$A : E : B$$

$2^{\text{nd}} \text{ O.}$        $2^{\text{nd}} \text{ O.}$        $2^{\text{nd}} \text{ O.}$

$$A_{ab} \quad E_{abcd} \quad B_{cd}$$

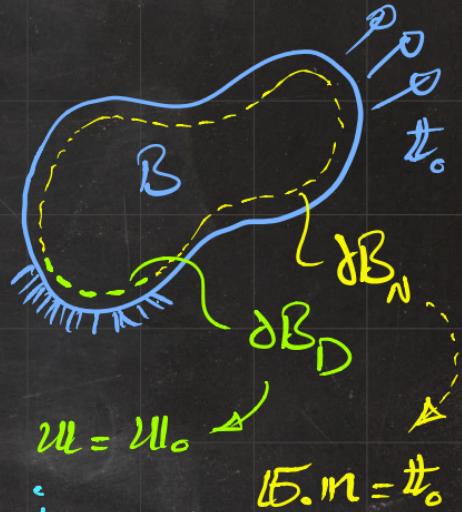
# FROM WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD(\omega)] : \underline{B} dA = \int_{\partial B_N} \omega \cdot \underline{t}_o dL$$

$$E_{abcd} \phi_a \otimes \phi_b \otimes \phi_c \otimes \phi_d$$

$$\int_B [\omega^j \otimes N^j_{,uu}] : E : [\underline{u}^i \otimes N^i_{,uu}] dA = \int_{\partial B_N} N^j \omega^j \cdot \underline{t}_o dL$$

$$\int_B [\omega^j]_a [N^j_{,uu}]_b E_{abcd} [\underline{u}^i]_c [N^i_{,uu}]_d dA = \int_{\partial B_N} N^j \hat{q} [\omega^j]_q [\underline{t}_o]_q dL$$

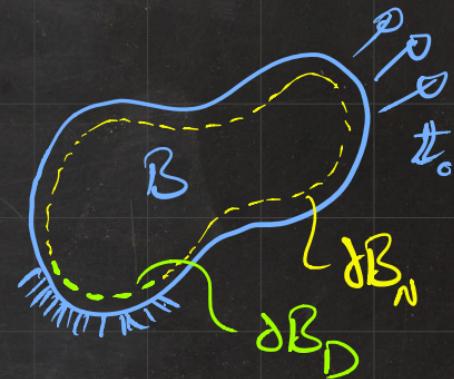


$$\underline{u} = \underline{u}_o$$

$$\underline{B} \cdot \underline{n} = \underline{t}_o$$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD \omega] : \underline{B} dA = \int_{\partial B_N} \omega \cdot \underline{t}_o dL$$

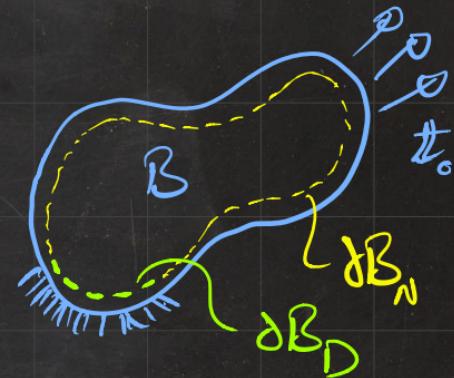


$$\int_B [\omega^j]_a [N^j_{,ab}]_b E_{abcd} [u^i]_c [N^i_{,cd}]_d dA = \int_{\partial B_N} N^j_q [\omega^j]_q [t^j]_q dL$$

$$[\omega^j]_a \int_B [N^j_{,ab}]_b E_{abcd} [N^i_{,cd}]_d dA [u^i]_c = [\omega^j]_q \int_{\partial B_N} N^j_q [t^j]_q dL$$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD(w)]^j B dA = \int_{\partial B_N} w \cdot t_o dL$$

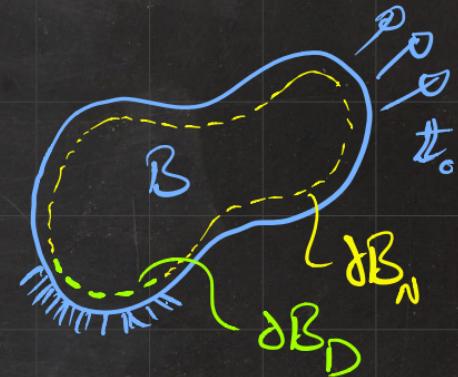


$$[w^j]_a \left[ \int_B [N^j_{,ac}]_b E_{abcd} [N^i_{,ad}]_d dA \right] \begin{matrix} \hat{[w^i]} \\ \hat{[u^i]} \end{matrix}_c = [w^j]_q \int_{\partial B_N} N^j_q [t^i]_q dL$$

\underbrace{\left[ K \right]\_{ac}^{ji}}\_{\substack{i,j \in \{1,2,\dots\} \\ a,c \in \{1,2\}}} \quad i,j \in \{1,2,\dots\} \\ a,c \in \{1,2\}

# FROM WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD(w)]^j B dA = \int_{\partial B_N} w \cdot t_o dL$$



$$[w^j]_a \left[ \int_B [N^j_{,ac}]_b E_{abcd} [N^i_{,ai}]_d dA \right] [\bar{u}^i]_c = [w^j]_q \int_{\partial B_N} N^j [t^i_q]_q dL$$

$$[w^j]_a [K^{ji}]_{ac} [\bar{u}^i]_c = [w^j]_q \int_{\partial B_N} N^j [t^i_q]_q dL$$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD(w)] : \underline{\underline{B}} dA = \int_{\partial B_N} w \cdot \underline{t}_o dL$$

$$[K]_{ac}^{ji} = \int_B [N_{,ac}^j]_b E_{abcd} [N_{,ac}^i]_d dA$$

$$[w^j]_a [K^{ji}]_{ac} [\underline{u}^i]_c = [w^j]_q \int_{\partial B_N} N^j [t_o]_q dL$$

$i, j \in \{1, \dots, NPE\}$

$[w^j]$  at node  $j$   
 $\xrightarrow{a}$  in direction  $a$   
 $\xrightarrow{x,y}$

$$[\omega^i]_1 = 1 \quad \text{AND ALL THE REST ARE ZERO}$$

$$\omega^1 = \begin{bmatrix} 1 \\ 0 \end{bmatrix} \quad \omega^2 = \begin{bmatrix} 0 \\ 1 \end{bmatrix} \quad \omega^3 = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \quad \dots$$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD(\omega)] : \underline{\underline{B}} dA = \int_{\partial B_N} \omega \cdot \underline{\underline{t}_o} dL$$

$$[K]_{ac}^{ji} = \int_B [N_{,ac}^j]_b E_{abcd} [N_{,ac}^i]_d dA$$

$$[\omega^j]_a [K^{ji}]_{ac} [\underline{\underline{u}}^i]_c = [\omega^j]_q \int_{\partial B_N} N^j [t_a]_q dL$$

$$[K^{1i}]_{1c} [\underline{\underline{u}}^i]_c = \int_{\partial B_N} N^1 [t_a]_1 dL$$

$\underline{\underline{F}}_x^1$        $\underline{\omega}^1 = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$   
 $\underline{\underline{F}}^1_1$        $\underline{\omega}^2 = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$   
 $\dots$        $\underline{\omega}^3 = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD \psi] : \underline{B} dA = \int_{\partial B_N} \psi \cdot \underline{\tau}_o dL$$

$$[K]_{ac}^{ji} = \int_B [N_{,ac}^j]_b E_{abcd} [N_{,ac}^i]_d dA$$

$$[\psi^j]_a [K]_{ac}^{ji} [\psi^i]_c = [\psi^j]_q \int_{\partial B_N} N^j [t_o]_q dL$$

$$\sum_i [K^{ii}]_{1c} [\psi^i]_c = [F^1]_1 + \sum_i \sum_c \dots$$

$$\psi^1 = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

$$\psi^2 = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$[K^{11}]_{11} [\psi^1]_1 + [K^{11}]_{12} [\psi^1]_2 + [K^{12}]_{11} [\psi^2]_1 + \dots$$

$$\psi^3 = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \dots$$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [ \text{GRAD}(\omega) ] : \underline{\underline{B}} \, dA = \int_{\partial B_N} \omega \cdot \underline{\underline{t}_o} \, dL$$

$$[K]_{ac}^{ji} = \int_B [N_{,ac}^j]_b E_{abcd} [N_{,ac}^i]_d \, dA$$

$$[\omega^j]_a [K]_{ac}^{ji} [\underline{\underline{u}}^i]_c = [\omega^j]_q \int_{\partial B_N} N^j [t_o]_q \, dL$$

$$\sum_i [K]_{1c}^{ii} [\underline{\underline{u}}^i]_c = [F^1]_1 \quad \text{and} \quad \sum_i \sum_c \dots$$

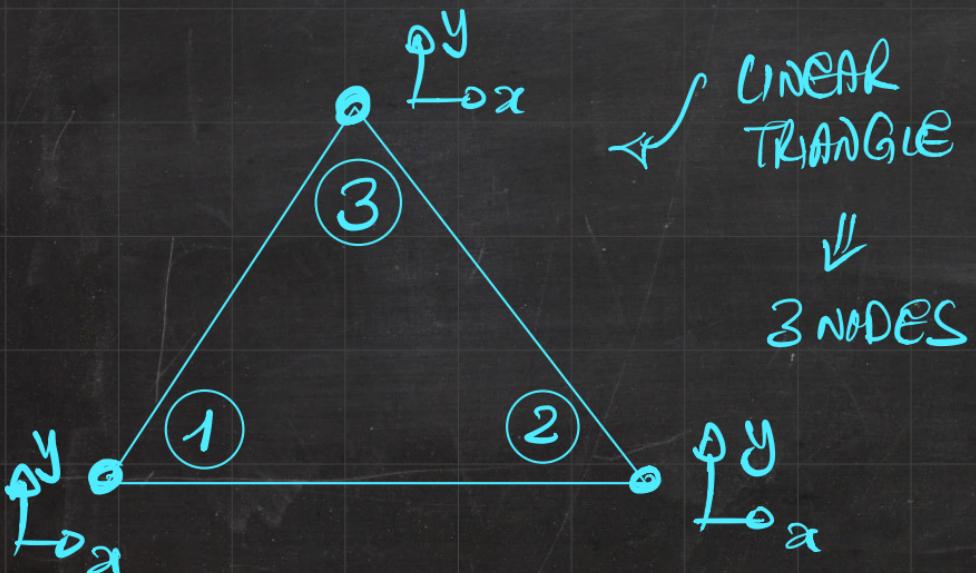
$$\begin{bmatrix} \dots & \dots & \dots \end{bmatrix} \begin{bmatrix} \vdots \\ \vdots \end{bmatrix} = [F^1]_1$$

$$\underline{\omega}^1 = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

$$\underline{\omega}^2 = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$\underline{\omega}^3 = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \dots$$

$$\begin{bmatrix} K_{11}^{11} & K_{12}^{11} & K_{11}^{12} & K_{12}^{12} & K_{11}^{13} & K_{12}^{13} \end{bmatrix}^T = F_1$$

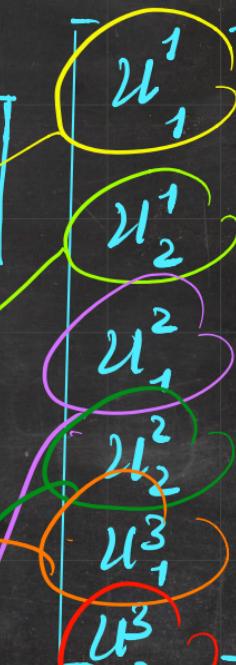
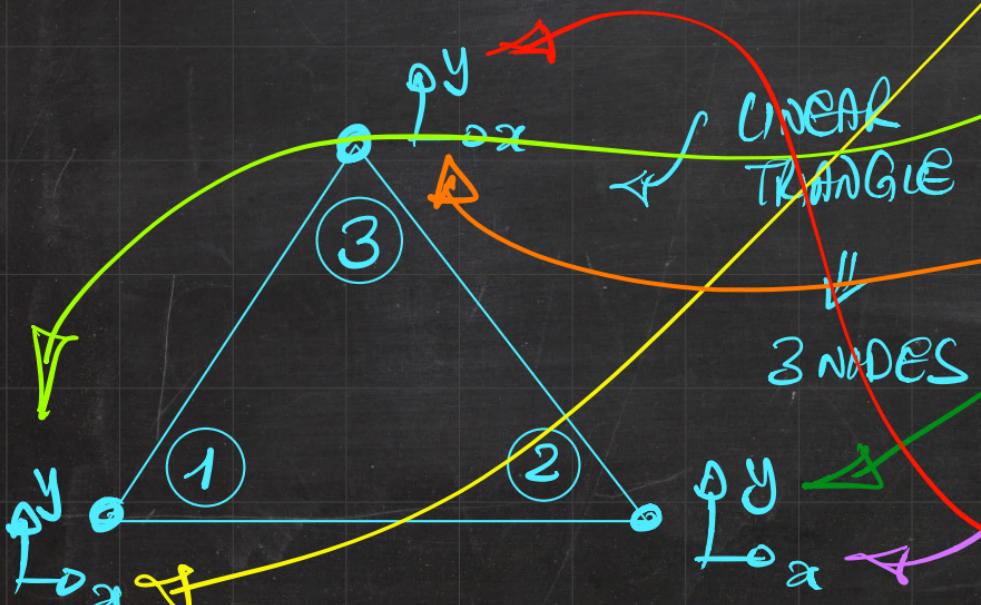


$$[K_{11}^{11} \quad K_{12}^{11}]$$

$$[K_{11}^{12} \quad K_{12}^{12}]$$

$$[K_{11}^{13} \quad K_{12}^{13}]$$

$$[K_{11}^{13} \quad K_{12}^{13}]$$



$$= F_{1, \frac{1}{x}}$$



Force of  
node 1  
in  
Direction 1



# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [ \text{GRAD}(\omega) ] : \underline{\underline{B}} \, dA = \int_{\partial B_N} \omega \cdot \underline{\underline{t}_o} \, dL$$

$$[K]_{ac}^{ji} = \int_B [N_{,ac}^j]_b E_{abcd} [N_{,ac}^i]_d \, dA$$

$$[\omega^j]_a [K]_{ac}^{ji} [\underline{\underline{u}}^i]_c = [\omega^j]_q \int_{\partial B_N} N^j [t_o]_q \, dL$$

$$\sum_i [K^{1i}]_{2c} [\underline{\underline{u}}^i]_c = [F^1]_2 \quad \text{and} \quad \sum_i \sum_c \dots$$

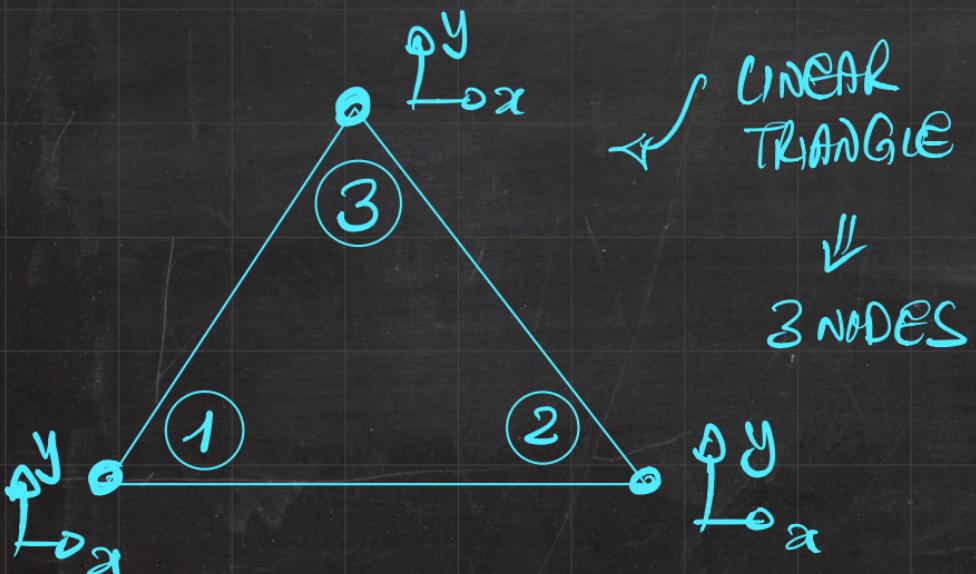
$$[ \dots \dots \dots ] \begin{bmatrix} \vdots \\ \vdots \end{bmatrix} = [F^1]_2$$

$$\underline{\omega}^1 = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$\underline{\omega}^2 = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\underline{\omega}^3 = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \dots$$

$$\begin{bmatrix} K_{21}^{11} & K_{22}^{11} & K_{21}^{12} & K_{22}^{12} & K_{21}^{13} & K_{22}^{13} \end{bmatrix} \begin{bmatrix} u_1^1 \\ u_1^2 \\ u_2^1 \\ u_2^2 \\ u_1^3 \\ u_2^3 \end{bmatrix} = F_2$$



$$\left[ \begin{array}{cc} K_{11}^{11} & K_{12}^{11} \\ K_{21}^{11} & K_{22}^{11} \end{array} \right] \left[ \begin{array}{cc} K_{11}^{12} & K_{12}^{12} \\ K_{21}^{12} & K_{22}^{12} \end{array} \right] \left[ \begin{array}{cc} K_{11}^{13} & K_{12}^{13} \\ K_{21}^{13} & K_{22}^{13} \end{array} \right] \left[ \begin{array}{c} u_1^1 \\ u_2^1 \\ u_1^2 \\ u_2^2 \\ u_1^3 \\ u_2^3 \end{array} \right] = \begin{array}{c} F_1 \\ F_2 \end{array}$$

$\Delta$   $\left[ \begin{array}{c} K^{11} \end{array} \right]$   $2 \times 2$

$\left[ \begin{array}{c} K \end{array} \right]_{6 \times 6}$  ✓

$$K^{11} = \left[ \begin{array}{cc} K_{xx}^{11} & K_{xy}^{11} \\ K_{yx}^{11} & K_{yy}^{11} \end{array} \right]$$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [GRAD \psi] : \underline{\underline{B}} dA = \int_{\partial B_N} \psi \cdot \underline{\underline{t}_o} dL$$

$$[K]_{ac}^{ji} = \int_B [N_{,ac}^j]_b E_{abcd} [N_{,ac}^i]_d dA$$

$$[\psi^j]_a [K]_{ac}^{ji} [\psi^i]_c = [\psi^j]_q \int_{\partial B_N} N^j [t_o]_q dL$$

$$\int [K]_{ac}^{ji} [\psi^i]_c = [F]_a^j$$

$[K]^{ji}$  STIFFNESS  
BETWEEN  
NODES  
 $\hookrightarrow$   
2x2  
matrix  
 $i \& j$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [ \text{GRAD}(\mathbf{u}) ] : \mathbf{B} \, dA = \int_{\partial B_N} \mathbf{u} \cdot \mathbf{t}_o \, dL$$

$$[K]_{ac}^{ji} = \int_B [N_{,ac}^j]_b E_{abcd} [N_{,ac}^i]_d \, da$$

$$[K]_{ac}^{ji} [u^i]_c = [F]^j_a$$

STIFFNESS  
 BETWEEN  
 NODES  
 $i \& j$   
 $\hookrightarrow$   
 $\leftarrow 2 \times 2$   
 matrix

$$[K]_{ac}^{ji} = \begin{bmatrix} K_{11}^{ji} & K_{12}^{ji} \\ K_{21}^{ji} & K_{22}^{ji} \end{bmatrix}$$

# From WEAK FORM TO APPROXIMATE FORM

$$\int_B [ \text{GRAD}(\mathbf{u}) ] : \mathbf{B} \, dA = \int_{\partial B_N} \mathbf{u} \cdot \mathbf{n}_o \, dL$$

$$[K]_{ac}^{ij} = \int_B [N_{,ac}]_b^i E_{abcd} [N_{,ac}]_d^j \, da$$

$$[K]_{ac}^{ij} [u]_c^j = [F]_a^i$$

$[K]$  STIFFNESS  
BETWEEN  
NODES  
 $i$  &  $j$

$\leftarrow$   
2x2  
matrix

$$\{i, j\} \in \{1, 2, \dots, NPE\}$$

$$[K] = \begin{bmatrix} K_{11}^{ij} & K_{12}^{ij} \\ K_{21}^{ij} & K_{22}^{ij} \end{bmatrix}$$

## APPROXIMATE FORM

$$[K^{ij}]_{ac} [u^j]_c = [F^i]_a$$

$$\rightarrow [K][u] = [F]$$

$\{i, j\} \in \{1, 2, \dots, NPE\}$

$$[K^{ij}]_{ac} = \int_B [N^i_{,ac}]^T_b E_{abcd} [N^j_{,cd}]_d dA$$

$[K^{ij}]$  STIFFNESS  
BETWEEN  
NODES  
 $i \& j$   
 $2 \times 2$   
matrix

$$[K^{ij}] = \begin{bmatrix} K_{11}^{ij} & K_{12}^{ij} \\ K_{21}^{ij} & K_{22}^{ij} \end{bmatrix}$$

UNDERSTANDING  $\left[ \begin{matrix} K^{ij} \\ \end{matrix} \right]_{ac} \rightarrow K^{ij}_{ac}$  ↗ SUPERSCRIPTS ↘ NODES  
↘ SUBSCRIPTS ↘ DIRECTIONS

$$[K^{ij}]_{ac}$$

$\hookrightarrow$  STIFFNESS BETWEEN  
NODES  $i$  &  $j$   
AND FOR  
DIRECTIONS  $a$  &  $c$   
RESPECTIVELY

$$[K_{ac}^{ij}] = \int_B [N_{ac}]^i_b E_{abcd} [N_{ad}]^j_d dA$$

① K ②  
oymo

$$\begin{bmatrix} K^{11} & K^{12} \\ K^{21} & K^{22} \end{bmatrix} \leftarrow K = \begin{bmatrix} K & -K \\ -K & K \end{bmatrix}$$

UNDERSTANDING  $[K^{ij}]_{ac}$



$$\begin{bmatrix} K^{11} & K^{12} \\ K^{21} & K^{22} \end{bmatrix} \leftarrow K = \begin{bmatrix} K & -K \\ -K & K \end{bmatrix}$$

$K^{11}$  NO STIFFNESS BETWEEN NODE 1&1

$$\begin{bmatrix} K^{11} & K^{12} \\ K^{21} & K^{22} \end{bmatrix} \begin{bmatrix} u^1 \\ u^2 \end{bmatrix} = \begin{bmatrix} F^1 \\ F^2 \end{bmatrix}$$

$$K^{11} = ?$$

$$K^{12} = ?$$

L STIFFNESS BETWEEN NODES 1&2

UNDERSTANDING  $[K^{ij}]_{ac}$   $\rightarrow$   $K^{ij}_{ac}$   $\rightarrow$  SUPERScripts  $\rightarrow$  NODES  
 $\rightarrow$  SUBSCRIPTS  $\rightarrow$  DIRECTIONS



$K^{11}$  NO STIFFNESS BETWEEN NODE 1&1

$$K^{ij} = \frac{\text{Change of Force @ NODE } i}{\text{Change of Disp. @ NODE } j}$$

$$\approx \frac{\delta F^i}{\delta u^j}$$

$$K^{12} = ?$$

$$\begin{bmatrix} K^{11} & K^{12} \\ K^{21} & K^{22} \end{bmatrix} \begin{bmatrix} u^1 \\ u^2 \end{bmatrix} = \begin{bmatrix} F^1 \\ F^2 \end{bmatrix}$$

STIFFNESS BETWEEN NODES 1&2

UNDERSTANDING  $[K^{ij}]_{ac}$   $\rightarrow$   $K^{ij}_{ac}$

$\nearrow$  superscripts  $\nwarrow$  nodes  
 $\searrow$  subscripts  $\nearrow$  directions

$$[K^{ij}]_{ac}$$

↳ STIFFNESS BETWEEN  
NODES  $i$  &  $j$   
AND FOR  
DIRECTIONS  $a$  &  $c$   
RESPECTIVELY

$$[K^{ij}]_{ac} = \int_B [N^i_{,ac}]^T_b E_{abcd} [N^j_{,ac}]_d dA$$

↳ STIFFNESS BETWEEN  
DIRECTION  $a$  OF NODE  $i$   
&  
DIRECTION  $c$  OF NODE  $j$

UNDERSTANDING  $[K^{ij}]_{ac}$   $\rightarrow K^{ij}_{ac}$

$$[K^{ij}]_{ac}$$

$$[K^{ij}]_{ac} = \int_B [N^i_{,ac}]^T_b E_{abcd} [N^j_{,bc}]_d dA$$

$$K^{ij}_{ac} = \frac{\delta F_a^i}{\delta u_c^j}$$

$\approx$  STIFFNESS BETWEEN  
DIRECTION a of NODE i  
&  
DIRECTION c of NODE j

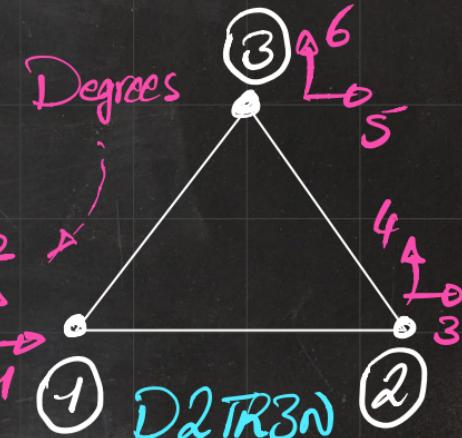
$$\Delta K = \begin{bmatrix} K^{11} & K^{12} & K^{13} \\ K^{21} & K^{22} & K^{23} \\ K^{31} & K^{32} & K^{33} \end{bmatrix}$$

6x6

$\Delta K^H = \frac{\delta F^1}{\delta U^1}$

D2TR3N

$$K^{ij} = \begin{bmatrix} K_{11}^{ij} & K_{12}^{ij} \\ K_{21}^{ij} & K_{22}^{ij} \end{bmatrix} = \begin{bmatrix} K_{xx}^{ij} & K_{xy}^{ij} \\ K_{yx}^{ij} & K_{yy}^{ij} \end{bmatrix}$$

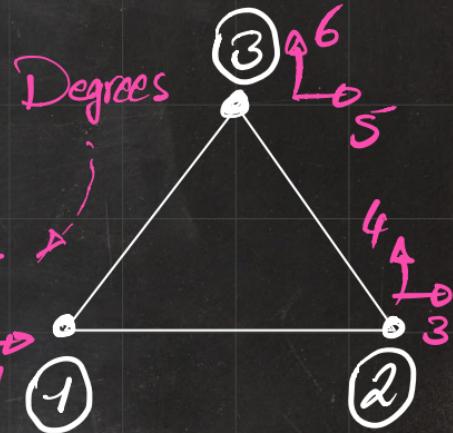


- 1 → NODE<sup>1</sup> X
- 2 → NODE<sup>1</sup> Y
- 3 → NODE<sup>2</sup> X
- 4 → NODE<sup>2</sup> Y
- 5 → NODE<sup>3</sup> X
- 6 → NODE<sup>3</sup> Y

$$\Delta K = \begin{bmatrix} K^{11} & K^{12} & K^{13} \\ K^{21} & K^{22} & K^{23} \\ K^{31} & K^{32} & K^{33} \end{bmatrix} \quad 6 \times 6$$

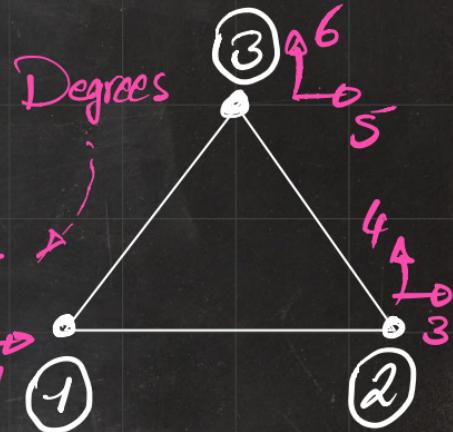
Non XPD  
t<sub>3</sub> t<sub>2</sub>

$$\Delta K = \begin{bmatrix} K_{11}^{11} & K_{12}^{11} & K_{11}^{12} & K_{12}^{12} & K_{11}^{13} & K_{12}^{13} \\ K_{21}^{11} & K_{22}^{11} & K_{11}^{21} & K_{21}^{21} & K_{11}^{22} & K_{21}^{22} \\ K_{11}^{21} & K_{21}^{21} & K_{11}^{22} & K_{21}^{22} & K_{11}^{23} & K_{21}^{23} \\ K_{21}^{22} & K_{22}^{22} & K_{11}^{31} & K_{21}^{31} & K_{11}^{32} & K_{21}^{32} \\ K_{21}^{31} & K_{22}^{31} & K_{21}^{32} & K_{22}^{32} & K_{11}^{33} & K_{12}^{33} \\ K_{21}^{32} & K_{22}^{32} & K_{21}^{33} & K_{22}^{33} & K_{21}^{33} & K_{22}^{33} \end{bmatrix}$$



$$\Delta K = \begin{bmatrix} K^{11} & K^{12} & K^{13} \\ K^{21} & K^{22} & K^{23} \\ K^{31} & K^{32} & K^{33} \end{bmatrix} \quad 6 \times 6$$

Non XPID  
t<sub>3</sub> t<sub>2</sub>



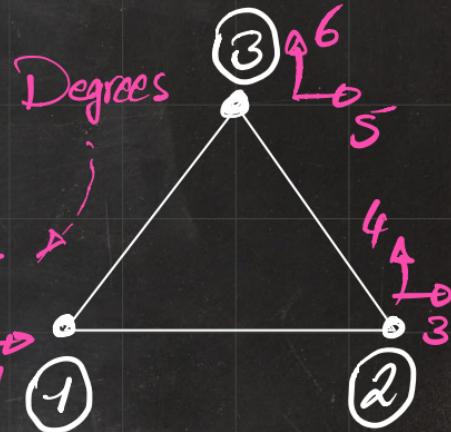
$$\Delta K = \begin{bmatrix} K_{11}^{11} & K_{12}^{11} & & & & \\ K_{21}^{11} & K_{22}^{11} & & & & \\ & & K_{11}^{12} & K_{12}^{12} & & \\ & & K_{21}^{12} & K_{22}^{12} & & \\ & & & & K_{11}^{13} & K_{12}^{13} \\ & & & & K_{21}^{13} & K_{22}^{13} \\ \hline K_{11}^{21} & K_{12}^{21} & & & & \\ K_{21}^{21} & K_{22}^{21} & & & & \\ & & K_{11}^{22} & K_{12}^{22} & & \\ & & K_{21}^{22} & K_{22}^{22} & & \\ & & & & K_{11}^{23} & K_{12}^{23} \\ & & & & K_{21}^{23} & K_{22}^{23} \\ \hline K_{11}^{31} & K_{12}^{31} & & & & \\ K_{21}^{31} & K_{22}^{31} & & & & \\ & & K_{11}^{32} & K_{12}^{32} & & \\ & & K_{21}^{32} & K_{22}^{32} & & \\ & & & & K_{11}^{33} & K_{12}^{33} \\ & & & & K_{21}^{33} & K_{22}^{33} \end{bmatrix}$$

Mapping of Nodes to Degrees of Freedom:

- 1 → NODE<sup>1</sup> X
- 2 → NODE<sup>1</sup> Y
- 3 → NODE<sup>2</sup> X
- 4 → NODE<sup>2</sup> Y
- 5 → NODE<sup>3</sup> X
- 6 → NODE<sup>3</sup> Y

$$\Delta K = \begin{bmatrix} K^{11} & K^{12} & K^{13} \\ K^{21} & K^{22} & K^{23} \\ K^{31} & K^{32} & K^{33} \end{bmatrix} \quad 6 \times 6$$

Non XPD  
t<sub>3</sub> t<sub>2</sub>



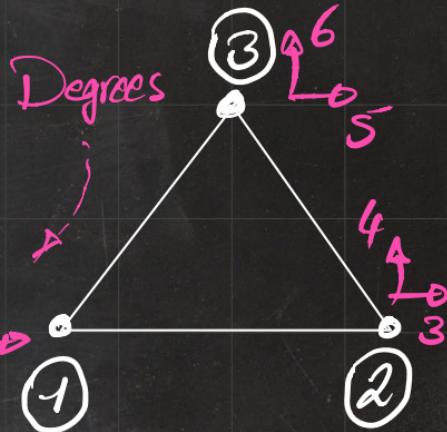
$$\Delta K = \begin{bmatrix} 1_x & 1_y & 2_x & 2_y & 3_x & 3_y \\ K_{11} & K_{12} & K_{13} & K_{14} & K_{15} & K_{16} \\ K_{21} & K_{22} & K_{23} & K_{24} & K_{25} & K_{26} \\ K_{31} & K_{32} & K_{33} & K_{34} & K_{35} & K_{36} \\ K_{41} & K_{42} & K_{43} & K_{44} & K_{45} & K_{46} \\ K_{51} & K_{52} & K_{53} & K_{54} & K_{55} & K_{56} \\ K_{61} & K_{62} & K_{63} & K_{64} & K_{65} & K_{66} \end{bmatrix}$$

Mapping of Nodes to Degrees of Freedom:

- 1 → NODE<sup>1</sup> X
- 2 → NODE<sup>1</sup> Y
- 3 → NODE<sup>2</sup> X
- 4 → NODE<sup>2</sup> Y
- 5 → NODE<sup>3</sup> X
- 6 → NODE<sup>3</sup> Y

$$\Delta K = \begin{bmatrix} K^{11} & K^{12} & K^{13} \\ K^{21} & K^{22} & K^{23} \\ K^{31} & K^{32} & K^{33} \end{bmatrix} \quad 6 \times 6$$

Non XPD  
t<sub>3</sub> t<sub>2</sub>



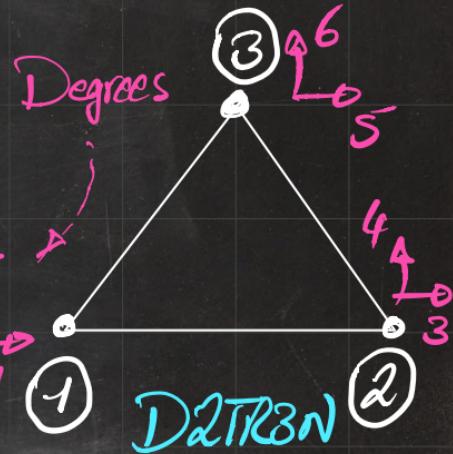
$$\Delta K = \begin{bmatrix} 1_x & 1_y & 2_x & 2_y & 3_x & 3_y \\ K_{11} & K_{12} & K_{13} & K_{14} & K_{15} & K_{16} \\ K_{21} & K_{22} & K_{23} & K_{24} & K_{25} & K_{26} \\ K_{31} & K_{32} & K_{33} & K_{34} & K_{35} & K_{36} \\ K_{41} & K_{42} & K_{43} & K_{44} & K_{45} & K_{46} \\ K_{51} & K_{52} & K_{53} & K_{54} & K_{55} & K_{56} \\ K_{61} & K_{62} & K_{63} & K_{64} & K_{65} & K_{66} \end{bmatrix}$$

Mapping from node numbers to degrees of freedom:

- 1 → NODE<sup>1</sup> X
- 1 → NODE<sup>1</sup> Y
- 2 → NODE<sup>2</sup> X
- 2 → NODE<sup>2</sup> Y
- 3 → NODE<sup>3</sup> X
- 3 → NODE<sup>3</sup> Y

$$\begin{matrix} \Delta \\ ||K|| \end{matrix} = \begin{bmatrix} ||K^{11}|| & ||K^{12}|| & ||K^{13}|| \\ ||K^{21}|| & ||K^{22}|| & ||K^{23}|| \\ ||K^{31}|| & ||K^{32}|| & ||K^{33}|| \end{bmatrix} \quad 6 \times 6$$

NonXPID  
 $t_3$     $t_2$



$$\begin{matrix} \Delta \\ ||K|| \end{matrix} = \begin{bmatrix} K_{11} & K_{12} & K_{13} & K_{14} & K_{15} & K_{16} \\ K_{21} & K_{22} & K_{23} & K_{24} & K_{25} & K_{26} \\ K_{31} & K_{32} & K_{33} & K_{34} & K_{35} & K_{36} \\ K_{41} & K_{42} & K_{43} & K_{44} & K_{45} & K_{46} \\ K_{51} & K_{52} & K_{53} & K_{54} & K_{55} & K_{56} \\ K_{61} & K_{62} & K_{63} & K_{64} & K_{65} & K_{66} \end{bmatrix}$$

DEGREES  
 1   2   3   4   5   6

1 → 1 mo NODE<sup>1</sup> X  
 2 → 2 mo NODE<sup>1</sup> Y  
 3 → 3 mo NODE<sup>2</sup> X  
 4 → 4 mo NODE<sup>2</sup> Y  
 5 → 5 mo NODE<sup>3</sup> X  
 6 → 6 mo NODE<sup>3</sup> Y

# ASSEMBLY

$$[K] = \begin{bmatrix} K_{11} & K_{12} & & & & \\ K_{21} & K_{22} & & & & \\ & & K_{33} & K_{34} & K_{35} & K_{36} \\ & & K_{43} & K_{44} & K_{45} & K_{46} \\ & & K_{53} & K_{54} & K_{55} & K_{56} \\ & & K_{63} & K_{64} & K_{65} & K_{66} \end{bmatrix}$$



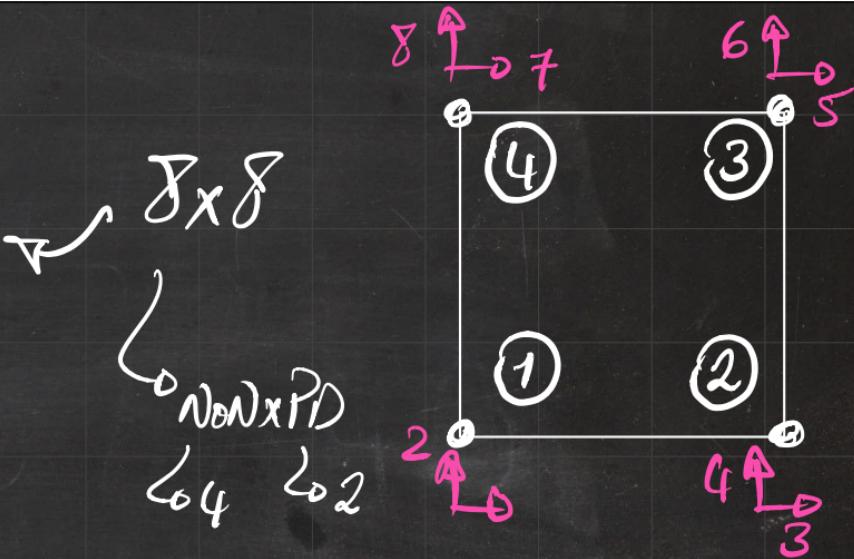
$$[K] = \frac{1}{\Delta} \begin{bmatrix} K_{11} & K_{12} & & & & \\ K_{21} & K_{22} & & & & \\ & & K_{33} & K_{34} & K_{35} & K_{36} \\ & & K_{43} & K_{44} & K_{45} & K_{46} \\ & & K_{53} & K_{54} & K_{55} & K_{56} \\ & & K_{63} & K_{64} & K_{65} & K_{66} \end{bmatrix}$$

⇒ Question  
ELEMENT STIFFNESS

$$\mathbb{K} = \begin{bmatrix} K^{11} & K^{12} & K^{13} & K^{14} \\ K^{21} & K^{22} & K^{23} & K^{24} \\ K^{31} & K^{32} & K^{33} & K^{34} \\ K^{41} & K^{42} & K^{43} & K^{44} \end{bmatrix}$$

D2Q4(4n)

$$\Delta \quad \mathbb{K}^{ij} = \begin{bmatrix} K_{11}^{ij} & K_{12}^{ij} \\ K_{21}^{ij} & K_{22}^{ij} \end{bmatrix} = \begin{bmatrix} K_{xx}^{ij} & K_{xy}^{ij} \\ K_{yx}^{ij} & K_{yy}^{ij} \end{bmatrix}$$



1, 2 → NODE<sup>1</sup><sub>x, y</sub>  
 3, 4 → NODE<sup>2</sup><sub>x, y</sub>  
 5, 6 → NODE<sup>3</sup><sub>x, y</sub>  
 7, 8 → NODE<sup>4</sup><sub>x, y</sub>

$$D2TR3N \curvearrowleft [K]_{6 \times 6}$$

$$D2TR6N \curvearrowleft [K]_{12 \times 12}$$

$$D2QU4N \curvearrowleft$$

$$D2QU8N \curvearrowleft [K]_{8 \times 8}$$

$$D2QU9N \curvearrowleft [K]_{16 \times 16}$$

$$\curvearrowleft [K]_{18 \times 18}$$

$$[K]_{ac}^{ij} = \int_B [N]_{ac}^i [N]_{cd}^j E_{abcd} dA$$

PROBLEMS TO ADDRESS

$\hookrightarrow$  INTEGRAL  $\hookrightarrow$  Gauss Quadrature Rule

$\hookrightarrow f(x) \approx x \rightarrow ?$

$\hookrightarrow E_{abcd} \approx ?$

$$D2TR3N \xrightarrow{\curvearrowleft} [K]_{6 \times 6} \quad E_{abcd} = \frac{E}{2[1+\nu]} [\delta_{ad}\delta_{bc} + \delta_{ac}\delta_{bd}]$$

$$D2TR6N \xrightarrow{\curvearrowleft} [K]_{12 \times 12}$$

$$D2QU4N \xrightarrow{\curvearrowleft}$$

$$D2QU8N \xrightarrow{\curvearrowleft} [K]_{8 \times 8}$$

$$D2QU9N \xrightarrow{\curvearrowleft} [K]_{16 \times 16}$$

$$\xrightarrow{\curvearrowleft} [K]_{18 \times 18}$$

$$[K]_{ac}^{ij} = \int_B [N]_{ac}^i [N]_{bd}^j E_{abcd}$$

$$+ \frac{EV}{1-\nu^2} \delta_{ab} \delta_{cd}$$

CONSTITUTIVE  
TENSOR

$\downarrow$   
4th.O.

$2 \times 2 \times 2 \times 2 = 16$   
COMPONENTS

Young's  
Modulus

$\nu$ : Poisson's  
Ratio

$\nu$ : nu

$\delta$ : Kronecker  
Delta

$i$   
 $j$

$a$   
 $b$

$c$   
 $d$

$$D2TR3N \quad \curvearrowleft [K]_{6 \times 6} \quad E_{abcd} = \frac{E}{2[1+\nu]} [\delta_{ad}\delta_{bc} + \delta_{ac}\delta_{bd}]$$

$$D2TR6N \quad \curvearrowleft [K]_{12 \times 12}$$

$$D2QU4N$$

$$D2QU8N$$

$$D2QU9N$$

$$E_{abcd} = \frac{E}{1-\nu^2} \delta_{ab}\delta_{cd}$$

$$E_{\substack{1111 \\ a'b'c'd'}} = \frac{E}{2[1+\nu]} [1 \times 1 + 1 \times 1]$$

$$+ \frac{EV}{1-\nu^2} 1 \times 1$$

$$[K]_{18 \times 18} \quad [K]_{ac}^{ij} = \int_B [N]_{,ai}^i [N]_{,aj}^j E_{abcd} [N]_{,aj}^d dA$$

$$D2TR3N \quad \curvearrowleft [K]_{6 \times 6} \quad E_{abcd} = \frac{E}{2[1+\nu]} [\delta_{ad}\delta_{bc} + \delta_{ac}\delta_{bd}]$$

$$D2TR6N \quad \curvearrowleft [K]_{12 \times 12} \quad V = \frac{\nu_{3D}}{1-\nu_{3D}} + \frac{EV}{1-\nu^2} \delta_{ab}\delta_{cd}$$

$$D2QU4N \quad \curvearrowleft [K]_{8 \times 8} \quad E_{\text{Plane Strain}} = \frac{E}{[1+\nu]} + \frac{EV}{1-\nu^2}$$

$$D2QU8N$$

$$[K]_{8 \times 8}$$

$$E_{\text{Plane Strain}} = \frac{E}{[1+\nu]} + \frac{EV}{1-\nu^2}$$

$$D2QU9N$$

$$[K]_{16 \times 16}$$

$$a' b' c' d' = \frac{E[1-\nu] + EV}{1-\nu^2} = \frac{E}{1-\nu^2}$$

$$[K]_{18 \times 18}$$

$$[K]_{ac}^{ij} = \int_B [N]_{ac}^i [N]_{bd}^j E_{abcd} [N]_{bd}^j dA$$

$$D2TR3N \quad \curvearrowleft \quad [K]_{6 \times 6} \quad E_{abcd} = \frac{E}{2[1+\nu]} [\delta_{ad}\delta_{bc} + \delta_{ac}\delta_{bd}]$$

$$D2TR6N \quad \curvearrowleft \quad [K]_{12 \times 12}$$

$$D2QU4N \quad \curvearrowleft$$

$$D2QU8N \quad \curvearrowleft \quad [K]_{8 \times 8}$$

$$D2QU9N \quad \curvearrowleft \quad [K]_{16 \times 16}$$

$$\begin{aligned} & \quad [K]_{18 \times 18} \quad [K]_{ac}^{ij} = \int_B [N]_{,ai}^i [N]_{,aj}^j E_{abcd} [N]_{,aj}^d dA \\ & \quad a \quad b \quad c \quad d \end{aligned}$$

$$\begin{aligned} E_{1,1,1,2} &= \frac{E}{2[1+\nu]} [0 \times 1 + 1 \times 0] \\ &+ \frac{E\nu}{1-\nu^2} 1 \times 0 \\ &= 0 \end{aligned}$$

$$[K_{ac}^{ij}] = \int_B [N_{,ac}]_b^i E_{abcd} [N_{,cd}]_d^j dA$$

$$E_{abcd} = \frac{E}{2[1+\nu]} [\delta_{ad}\delta_{bc} + \delta_{ac}\delta_{bd}] + \frac{EV}{1-\nu^2} \delta_{ab}\delta_{cd}$$

$$E_{1111} = \frac{E}{[1-\nu^2]}$$

$$E_{1112} = 0$$

$$E_{1121} = 0$$

$$E_{1122} = \frac{E}{[1-\nu^2]}$$

$$E_{1211} = 0$$

$$E_{1212} = \frac{E}{2[1+\nu]}$$

$$E_{1221} = \frac{E}{2[1+\nu]}$$

$$E_{1222} = 0$$

$$E_{2111} = 0$$

$$E_{2112} = \frac{E}{2[1+\nu]}$$

$$E_{2121} = \frac{E}{2[1+\nu]}$$

$$E_{2122} = 0$$

$$E_{2211} = \frac{E}{[1-\nu^2]}$$

$$E_{2212} = 0$$

$$E_{2221} = 0$$

$$E_{2222} = \frac{E}{[1-\nu^2]}$$

From STRONG FORM TO ELEMENT STIFFNESS  $\rightarrow$  IN PHYSICAL SPACE now

$$EA u'' = 0 \quad \text{SUBJECT TO BCs}$$



$$EA \begin{bmatrix} \int_L N^1' N^1' dx & \int_L N^1' N^2' dx \\ \int_L N^2' N^1' dx & \int_L N^2' N^2' dx \end{bmatrix} \begin{bmatrix} u^1 \\ u^2 \end{bmatrix} = \begin{bmatrix} F^1 \\ F^2 \end{bmatrix}$$

$$\begin{bmatrix} K^{11} & K^{12} \\ K^{21} & K^{22} \end{bmatrix} \begin{bmatrix} u^1 \\ u^2 \end{bmatrix} = \begin{bmatrix} F^1 \\ F^2 \end{bmatrix} \quad \text{and} \quad K^{ij} = EA \int_L N^i' N^j' dx$$

$$K^{ij} = EA \int_L n^i' n^j' dx \quad \xrightarrow{\text{PHYSICAL}} \text{RECALL:}$$

$$= EA \int_{-1}^1 \frac{\partial N^i}{\partial \xi} \frac{\partial N^j}{\partial \xi} \bar{J}^{-1} d\xi \quad \xrightarrow{\text{NATURAL}}$$

$$\int_{-1}^1 g(\xi) d\xi = \sum_{GP=1}^{GPE} g(\xi) \alpha_{GP}$$

$\leftarrow$  Loop over GP

$$= EA \sum_{GP=1}^{GPE} \left\{ \left[ \frac{\partial N^i}{\partial \xi} \quad \frac{\partial N^j}{\partial \xi} \quad \bar{J}^{-1} \right] \Big|_{GP} \times \alpha_{GP} \right\} \quad \vdots \quad \text{END}$$

)  
eg.

WHAT YOU  
SEE IN THE  
CODE !

{ For  $GP=1: GPE$   
in  
MATLAB  
End

$$[K^{ij}]_{ac} = \int_B [N_{,ac}]_b^i E_{abcd} [N_{,ac}]_d^j dA$$

$$\alpha = \alpha(\xi, \eta)$$

$$\alpha = \alpha(\xi) \quad \gamma = \gamma(\xi, \eta)$$

$$N_{,ac}^i$$

$\rightarrow$

$$\begin{bmatrix} \frac{\partial N^i}{\partial x} \\ \frac{\partial N^i}{\partial y} \end{bmatrix}$$

$$\frac{\partial N^i}{\partial x} = \frac{\partial N^i}{\partial \xi} \frac{\partial \xi}{\partial x} + \frac{\partial N^i}{\partial \eta} \frac{\partial \eta}{\partial x}$$

$$\xi = \xi(x, y)$$

$$\frac{\partial N^i}{\partial y} = \frac{\partial N^i}{\partial \xi} \frac{\partial \xi}{\partial y} + \frac{\partial N^i}{\partial \eta} \frac{\partial \eta}{\partial y}$$

$$\eta = \eta(x, y)$$



$\uparrow$   
 $\downarrow$

$$\xi = \xi(x)$$

$$[K_{ac}^{ij}] = \int_B [N_{,ac}]_b^i E_{abcd} [N_{,\alpha}]_d^j dA$$

$$\alpha = \alpha(\xi, \eta)$$

$$\alpha = \alpha(\xi) \quad \gamma = \gamma(\xi, \eta)$$

$$\begin{bmatrix} \frac{\partial N^i}{\partial x} \\ \frac{\partial N^i}{\partial y} \end{bmatrix} = \begin{bmatrix} \frac{\partial \xi}{\partial x} & \frac{\partial \eta}{\partial x} \\ \frac{\partial \xi}{\partial y} & \frac{\partial \eta}{\partial y} \end{bmatrix} \begin{bmatrix} \frac{\partial N^i}{\partial \xi} \\ \frac{\partial N^i}{\partial \eta} \end{bmatrix}$$

$$\frac{\partial N^i}{\partial x} = \frac{\partial N^i}{\partial \xi} \frac{\partial \xi}{\partial x} + \frac{\partial N^i}{\partial \eta} \frac{\partial \eta}{\partial x}$$

$$\frac{\partial N^i}{\partial y} = \frac{\partial N^i}{\partial \xi} \frac{\partial \xi}{\partial y} + \frac{\partial N^i}{\partial \eta} \frac{\partial \eta}{\partial y}$$

$$\xi = \xi(x, y)$$

$$\eta = \eta(x, y)$$

$$\xi = \xi(\alpha)$$

$$[K^{ij}]_{ac} = \int_B [N_{,ac}]_b^i E_{abcd} [N_{,dc}]_d^j dA$$

$$\alpha = \alpha(\xi, \eta)$$

$$\begin{bmatrix} \frac{\partial N^i}{\partial x} \\ \frac{\partial N^i}{\partial y} \end{bmatrix} = \begin{bmatrix} \frac{\partial \xi}{\partial x} & \frac{\partial \eta}{\partial x} \\ \frac{\partial \xi}{\partial y} & \frac{\partial \eta}{\partial y} \end{bmatrix} \begin{bmatrix} \frac{\partial N^i}{\partial \xi} \\ \frac{\partial N^i}{\partial \eta} \end{bmatrix}$$

$\underbrace{N^i_{,\alpha}}$        $\underbrace{N^i_{,\xi}}$

$$\frac{\partial N^i}{\partial x} \quad N^i_{,\alpha}$$

$$\xi = \xi(x, y)$$

$$\frac{\partial N^i}{\partial \xi} \quad N^i_{,\xi}$$

$$\eta = \eta(x, y)$$

$$\xi = \xi(\alpha) \quad \alpha$$

$$[K_{ac}^{ij}] = \int_B [N_{,ac}]_b^i E_{abcd} [N_{,a}]_d^j dA$$

$$\alpha = \alpha(\xi, \eta)$$

$$\alpha = \alpha(\xi) \quad \eta = \eta(\xi, \eta)$$

$$\begin{bmatrix} \frac{\partial N^i}{\partial x} \\ \frac{\partial N^i}{\partial y} \end{bmatrix} = \begin{bmatrix} \frac{\partial \xi}{\partial x} & \frac{\partial \eta}{\partial x} \\ \frac{\partial \xi}{\partial y} & \frac{\partial \eta}{\partial y} \end{bmatrix} \begin{bmatrix} \frac{\partial N^i}{\partial \xi} \\ \frac{\partial N^i}{\partial \eta} \end{bmatrix}$$

$$\tilde{N}_{,\alpha}^i$$

$$\tilde{N}_{,\xi}^i$$

$$N_{,\alpha}^i = \begin{bmatrix} \dots \\ N_{,\xi}^i \end{bmatrix}$$

$$\xi = \xi(x, y)$$

$$\eta = \eta(x, y)$$

$$\xi = \xi(\alpha)$$

$$N_{\alpha}^i$$

$$[K^{ij}]_{ac} = \int_B [N_{\alpha}^i]_b E_{abcd} [N_{\alpha}^j]_d dA$$

$$\alpha = \alpha(\xi, \eta)$$

$$\begin{bmatrix} \frac{\partial \xi}{\partial x} & \frac{\partial \xi}{\partial y} \\ \frac{\partial \eta}{\partial x} & \frac{\partial \eta}{\partial y} \end{bmatrix} \leftarrow J \quad \frac{\partial \xi}{\partial \alpha} \begin{bmatrix} \frac{\partial u}{\partial \nu} \end{bmatrix}_{\alpha\beta} = \frac{\partial u_{\alpha}}{\partial \nu_{\beta}}$$

$\alpha = \alpha(\xi) \quad \eta = \eta(\xi, \gamma)$

$$N_{\alpha}^i = \begin{bmatrix} \dots \\ \dots \\ N_{\alpha}^i \end{bmatrix}$$

$\xi = \xi(x, y) \quad \gamma = \gamma(x, y)$

$$J \not\rightarrow \frac{\partial \alpha}{\partial \xi} \begin{bmatrix} \frac{\partial x}{\partial \xi} & \frac{\partial x}{\partial \eta} \\ \frac{\partial y}{\partial \xi} & \frac{\partial y}{\partial \eta} \end{bmatrix} \quad \begin{bmatrix} \frac{\partial \xi}{\partial x} & \frac{\partial \eta}{\partial x} \\ \frac{\partial \xi}{\partial y} & \frac{\partial \eta}{\partial y} \end{bmatrix}$$

$\xi = \xi(\alpha) \quad \gamma = \gamma(\alpha)$

$$N_{,x\zeta}^i$$

$$[K^{ij}]_{ac} = \int_B [N_{,ac}]_b E_{abcd} [N_{,d\zeta}]_c dA$$

$$\chi = \chi(\xi, \eta)$$

$$\begin{bmatrix} \frac{\partial \xi}{\partial x} & \frac{\partial \xi}{\partial y} \\ \frac{\partial \eta}{\partial x} & \frac{\partial \eta}{\partial y} \end{bmatrix} \Leftarrow \bar{J}$$

$$\bar{J}^{-1} = \frac{\partial \chi}{\partial \xi}$$

$$N_{,\chi\zeta}^i = \begin{bmatrix} \dots \\ \dots \\ \end{bmatrix} N_{,\xi\zeta}^i$$

$$y = \eta(\xi, \eta)$$

$\uparrow$   
 $\downarrow$

$$\xi = \xi(x, y)$$

$$\begin{bmatrix} \frac{\partial \xi}{\partial x} & \frac{\partial \eta}{\partial x} \\ \frac{\partial \xi}{\partial y} & \frac{\partial \eta}{\partial y} \end{bmatrix} = \bar{J}^{-1}$$

$$\eta = \eta(x, y)$$

$$\xi = \xi(\chi)$$

$$[K_{ac}^{ij}] = \int_B [N_{,ac}]_b^i E_{abcd} [N_{,d}]_d^j dA$$

$$[K_{ac}^{ij}] = \int_B [\bar{J} \cdot N_{,\xi}^i]_b E_{abcd} [\bar{J} \cdot N_{,\xi}^j]_d dA$$

$$\bar{J} = \frac{\partial \alpha}{\partial \xi} \quad \text{so} \quad \alpha = \alpha(\xi) \quad \text{and} \quad \alpha = N^s \alpha^s \\ \text{so } N^s(\xi, \eta)$$

$$[K_{ac}^{ij}] = \int_B [J^t \cdot N_{,\xi}^{ij}]_b E_{abcd} [\bar{J}^t \cdot N_{,\xi}^{jd}]_d dA$$

$$J_{11} = \left[ \frac{\partial x}{\partial \xi} \right]_{11} = \frac{\partial x}{\partial \xi} = \frac{\partial}{\partial \xi} (N^s x^s) = x^s \frac{\partial N^s}{\partial \xi}$$

$$J = \frac{\partial x}{\partial \xi} \quad \text{so} \quad x = x(\xi) \quad \text{and} \quad x = N^s x^s \\ \hookrightarrow N^s(\xi, \eta)$$

$$[K_{ac}^{ij}] = \int_B [J^t \cdot N_{,g}^{ij}]_b E_{abcd} [\bar{J}^t \cdot N_{,g}^{jd}]_d dA$$

$$J_{11} = \left[ \frac{\partial x}{\partial \xi} \right]_{11} = \frac{\partial x}{\partial \xi} = \frac{\partial}{\partial \xi} (N^s x^s) = x^s \frac{\partial N^s}{\partial \xi}$$

$$= x^1 \frac{\partial N^1}{\partial \xi} + x^2 \frac{\partial N^2}{\partial \xi} + \dots + x^{NPE} \frac{\partial N^{NPE}}{\partial \xi}$$

$$[K_{ac}^{ij}] = \int_B [J^t \cdot N_{,g}^{ij}]_b E_{abcd} [J^t \cdot N_{,g}^{jd}]_d dA$$

$$J_{11} = \frac{\partial x}{\partial \xi} = x^1 \frac{\partial N^1}{\partial \xi} + x^2 \frac{\partial N^2}{\partial \xi} + \dots + x^{NPE} \frac{\partial N^{NPE}}{\partial \xi}$$

$$J_{12} = \frac{\partial x}{\partial \eta} = x^1 \frac{\partial N^1}{\partial \eta} + x^2 \frac{\partial N^2}{\partial \eta} + \dots + x^{NPE} \frac{\partial N^{NPE}}{\partial \eta}$$

$$J_{21} = \frac{\partial y}{\partial \xi} = y^1 \frac{\partial N^1}{\partial \xi} + y^2 \frac{\partial N^2}{\partial \xi} + \dots + y^{NPE} \frac{\partial N^{NPE}}{\partial \xi}$$

$$J_{22} = \frac{\partial y}{\partial \eta} = y^1 \frac{\partial N^1}{\partial \eta} + y^2 \frac{\partial N^2}{\partial \eta} + \dots + y^{NPE} \frac{\partial N^{NPE}}{\partial \eta}$$

$$[K_{ac}^{ij}] = \int_B [J^t \cdot N_{,g}^{ij}]_b E_{abcd} [\bar{J}^t \cdot N_{,g}^{jd}]_d dA$$

$$\begin{bmatrix} J_{11} & J_{21} \\ J_{12} & J_{22} \end{bmatrix} = \underbrace{\begin{bmatrix} \frac{\partial N^1}{\partial \xi} & \frac{\partial N^2}{\partial \xi} & \dots & \frac{\partial N^{NPE}}{\partial \xi} \\ \frac{\partial N^1}{\partial \eta} & \frac{\partial N^2}{\partial \eta} & \dots & \frac{\partial N^{NPE}}{\partial \eta} \end{bmatrix}}_{J^t} \underbrace{\begin{bmatrix} x^1 & y^1 \\ x^2 & y^2 \\ \vdots & \vdots \\ x^{NPE} & y^{NPE} \end{bmatrix}}_{2 \times NPE} \underbrace{\begin{bmatrix} x^1 & y^1 \\ x^2 & y^2 \\ \vdots & \vdots \\ x^{NPE} & y^{NPE} \end{bmatrix}}_{NPE \times 2}$$

$$K_{ac}^{ij} = \sum_{GP=1}^{GPE} \left[ \bar{J}^t \cdot N_{,x}^{i,j} \right]_b \bar{E}_{abcd} \left[ \bar{J}^t \cdot N_{,x}^{i,j} \right]_b \text{Dot} J \times \alpha \times \frac{1}{2}$$

JACOBIAN  $\frac{\partial \mathbf{x}}{\partial \xi}$

$$\bar{J} = \begin{bmatrix} x^1_{\text{node}} & x^{\text{NPE}} \\ y^1_{\text{node}} & y^{\text{NPE}} \end{bmatrix} \begin{bmatrix} N_{,x}^1 & N_{,y}^1 \\ \vdots & \vdots \\ N_{,x}^{\text{NPE}} & N_{,y}^{\text{NPE}} \end{bmatrix}$$

IF TRIANGLE

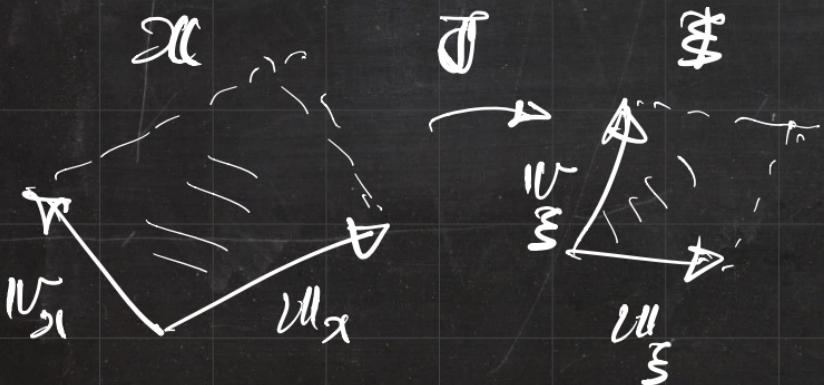
$$\bar{E}_{abcd} = \frac{E}{2[1+\nu]} [\delta_{ad}\delta_{bc} + \delta_{ac}\delta_{bd}] + \frac{EV}{1-\nu^2} \delta_{ab}\delta_{cd}$$

$$\left\{ \begin{array}{llll} E_{1111} = \frac{E}{[1-\nu^2]} & E_{1112} = 0 & E_{1121} = 0 & E_{1122} = \frac{E}{[1-\nu^2]} \\ E_{1211} = 0 & E_{1212} = \frac{E}{2[1+\nu]} & E_{1221} = \frac{E}{2[1+\nu]} & E_{1222} = 0 \\ E_{2111} = 0 & E_{2112} = \frac{E}{2[1+\nu]} & E_{2121} = \frac{E}{2[1+\nu]} & E_{2122} = 0 \\ E_{2211} = \frac{E}{[1-\nu^2]} & E_{2212} = 0 & E_{2221} = 0 & E_{2222} = \frac{E}{[1-\nu^2]} \end{array} \right.$$

$$dA_x = \text{Det } J \underbrace{dA_{\xi}}$$

$$\begin{vmatrix} i & j & k \\ x & x & x \\ x & x & x \end{vmatrix}$$

Exercise  $\rightarrow J_{11}J_{22} - J_{12}J_{21} dA_x = |w_x \times w_{21}|$



$$dA_{\xi} = |w_{\xi} \times w_{\bar{\xi}}|$$

$$dA_x = \text{Det } J \text{ } dA_{\xi}$$

$$\begin{matrix} 25/4 \\ \square \\ \delta \\ \delta \\ 25 \end{matrix}$$

